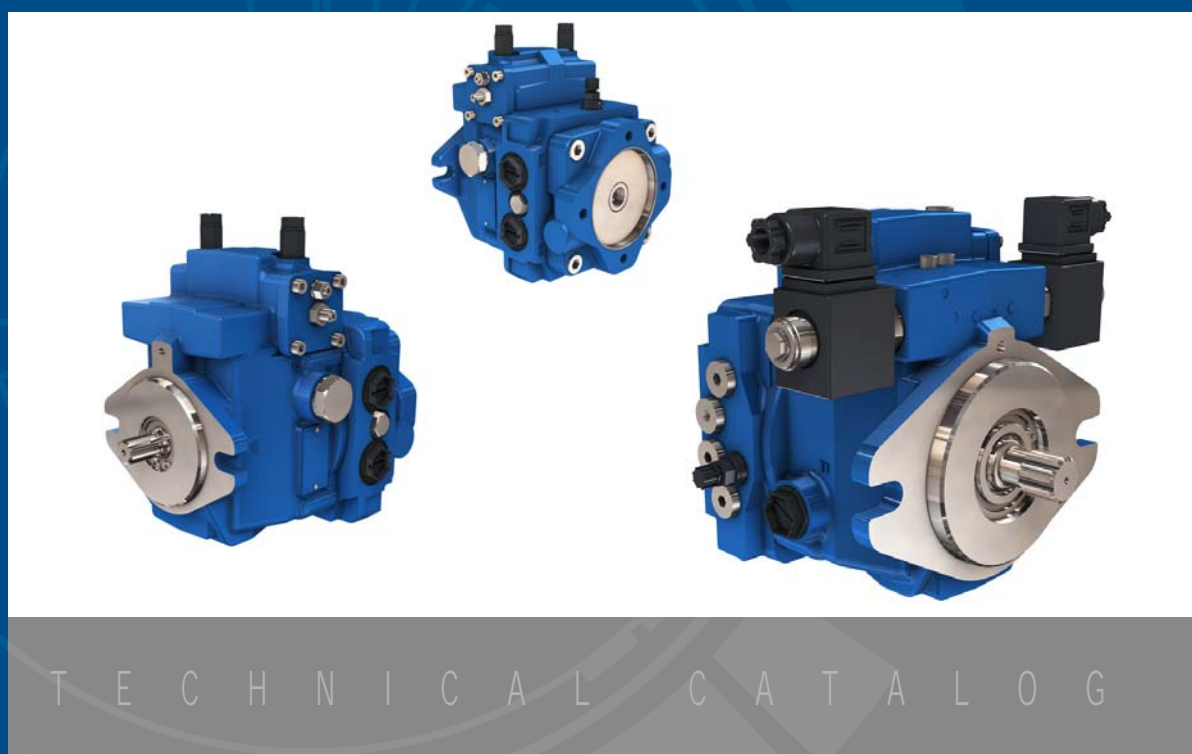


PM10

VARIABLE DISPLACEMENT PUMP
CLOSED LOOP CIRCUIT



T E C H N I C A L C A T A L O G



OVERVIEW

PM10 is a variable displacement, axial piston pump, with swashplate system, for closed loop hydrostatic transmissions.

It provides a continuously variable flow rate between zero and maximum in forward and reverse direction. Flow rate is proportional to rotation speed and swashplate angle.

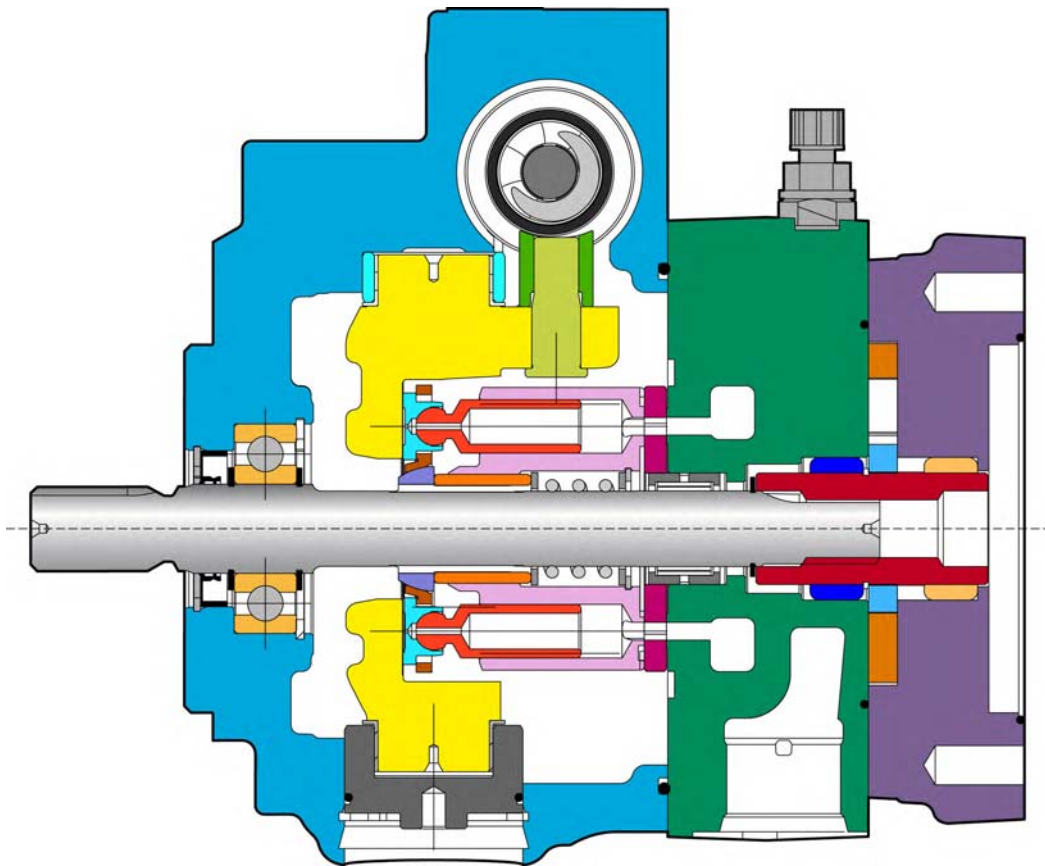
It can feature a charge pump to keep the circuit pressurised. This avoids risk of cavitations and ensures a good performance of the transmission.

It offers several types of control: direct mechanical, servo hydraulic, servo mechanical, electrical, proportional electrical and automotive.

It is equipped with high pressure relief valves and can be delivered with auxiliary gear pumps.

It is available in single or tandem versions.

As options, PM10 can be featured with flushing valve, pressure cut-off, filter on charge pressure line and safety devices to ensure safe operation of the machine.



		PM10-07	PM10-09	PM10-11	PM10-13	PM10-14	PM10-16	PM10-18	PM10-21
Displacement	cm ³ /rev [in ³ /rev.]	7,08 [0.43]	9,08 [0.55]	11,83 [0.72]	13,89 [0.85]	14,32 [0.87]	16,80 [1.03]	17,85 [1.09]	20,40 [1.24]
Theoretical Flow at rated speed	L/min [GPM]	25,5 [6.74]	32,7 [8.64]	42,6 [11.25]	50,0 [13.21]	51,6 [13.63]	59 [15.59]	64,3 [16.99]	73 [19.28]
Rated speed	rpm	3 600							
Rated pressure	bar [PSI]	210 [3 045]							
Max. pressure	bar [PSI]	350 [5 076]							300 [4 351]
Mounting flange		SAE A, SAE B							
Controls		Direct mechanical, servo hydraulic with or without feedback, servo mechanical, electrical, electrico-proportional with or without feedback, automotive							
Mass	kg [lb]	from 16,3 [35.9] to 18,8 [41.4]							
Rotation		Clockwise or Counterclockwise							



CONTENT



MODEL CODE 4

Model
Code

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Controls

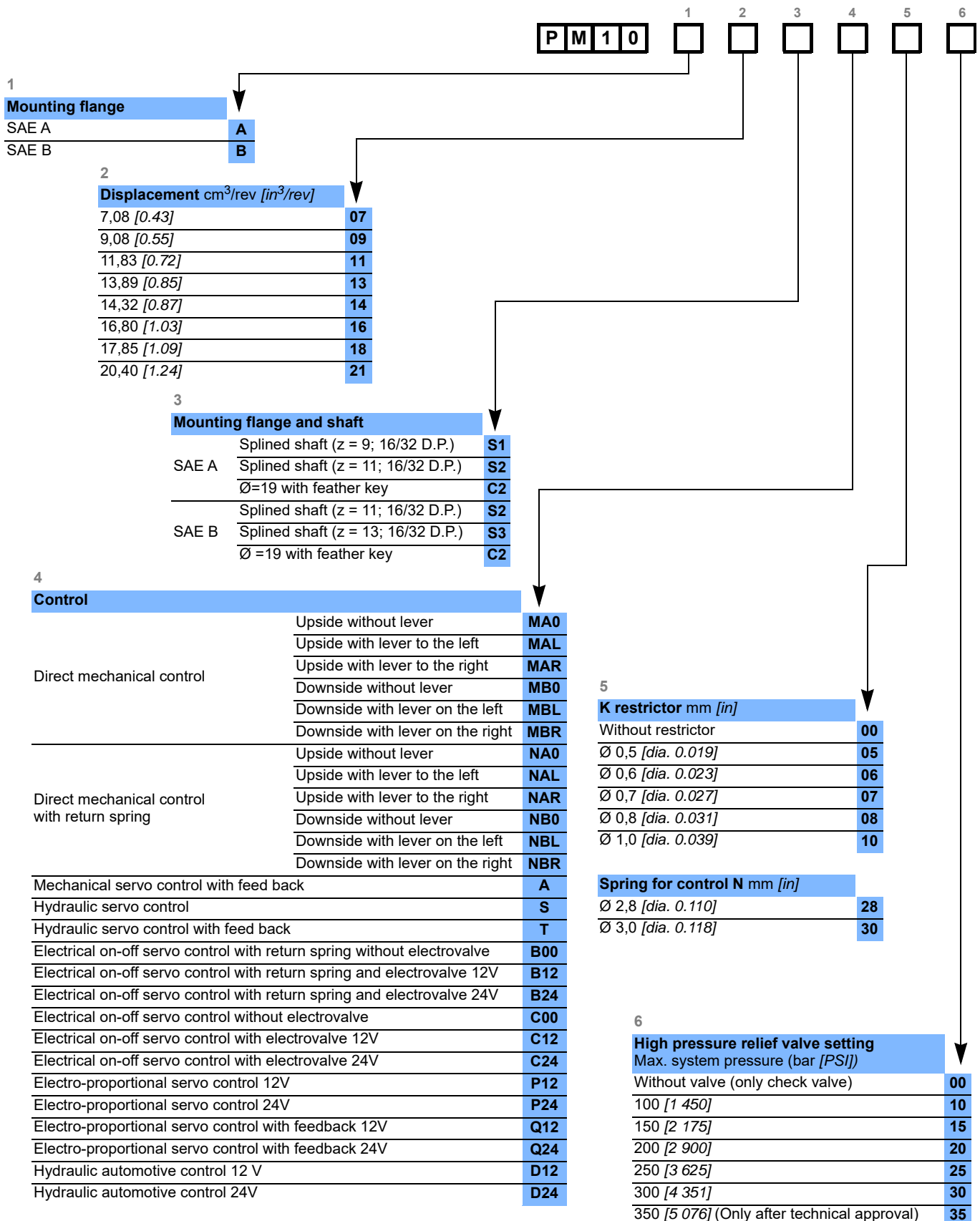
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Options



MODEL





CODE

7 8 9 10 11 12

7	Rotation
Clockwise	R
Counter clockwise	L

8	Charge relief valve setting bar [PSI]
Without charge relief valve	00
Setting 10 [145] (standard for M and N controls)	10
Setting 20 [290] (standard for servo control)	20

9	Charge pump displacement cm^3/rev [in^3/rev]
Without charge pump	00
4,9 [0.30] (For auxiliary mounting pad F and G)	05
7,0 [0.43] (Only for auxiliary mounting pad A)	07
9,0 [0.55] (For tandem pump with auxiliary mounting pad A and single pump)	08

10	Auxiliary mounting pad
Without auxiliary mounting pad	S
SAE A flange (z=9, 16/32 DP)	A
SAE A flange (z=11, 16/32 DP)	E
German group 1 flange	F
German group 2 flange	G

11	Gear pump cm^3/rev [$\text{cu.in}/\text{rev}$]
Without gear pump	00
1,6 [0.10]	01
2,0 [0.12]	02
3,2 [0.19]	03
4,2 [0.26]	04
5,0 [0.30]	05
6,3 [0.38]	06
4,5 [0.27]	04
6,0 [0.37]	06
8,5 [0.52]	08
11,0 [0.67]	11
14,5 [0.88]	14
17,0 [1.04]	17
SAE A flange (If digit 10 = A)	20

12

Options

Without option	00
Roller bearing	CR
Fluorinated elastomer seals	EV
UNF Threads	FU
Filter on pressure line without clogging indicator	F0
Filter on pressure line with clogging indicator	F2
External connections for filter	F3
Mechanical inching for control D	IC
Hydraulic inching for control D	HI
Neutral position switch (Only with control A)	MI
Safety valve (For controls A and S)	VPU
Flushing valve	VS
Finishing coat	PA
Customized identification plate	DP
Antistall valve	SD
Pressure cut-off valve	LP
Special version	ES

A screw by-pass is present at the base of all PM10 pumps.



In case of request for a combination of several options, please contact your Poclain Hydraulics application engineer for further information.



In case of specific control request (constant flow, antistall effect, ...) that require dedicated valve plate timing, please contact your Poclain Hydraulics application engineer for further information.

Model
CodeTechnical
specificationsOperating
ParametersSystem design
Parameters

Features

Controls

Options



TECHNICAL SPECIFICATIONS

Features

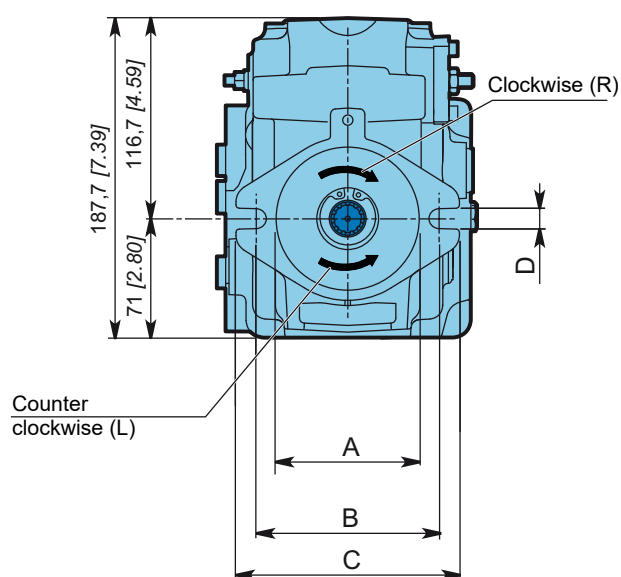
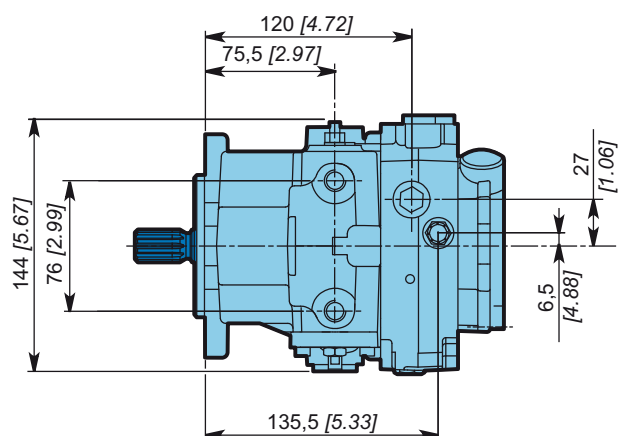
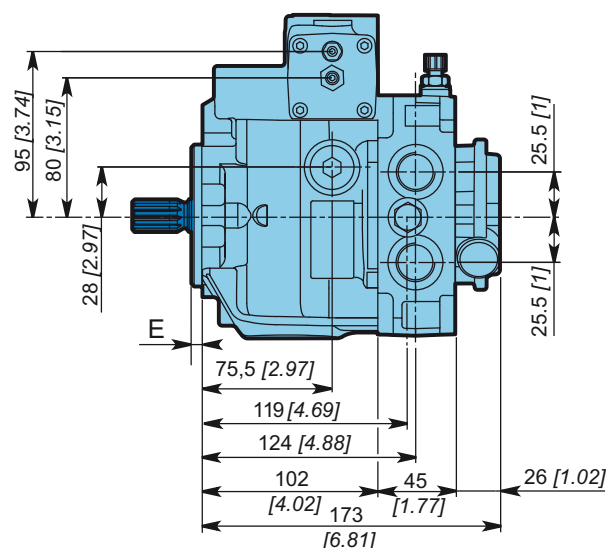
		PM10-07	PM10-09	PM10-11	PM10-13	PM10-14	PM10-16	PM10-18	PM10-21
Displacement	cm ³ /rev [in ³ /rev.]	7,08 [0.43]	9,08 [0.55]	11,83 [0.72]	13,89 [0.85]	14,32 [0.87]	16,80 [1.03]	17,85 [1.09]	20,40 [1.24]
Theoretical flow at rated speed (3600 rpm)	L/min [GPM]	25,5 [6.74]	32,7 [8.64]	42,6 [11.25]	50 [13.21]	51,6 [13.63]	59 [15.59]	64,3 [16.99]	73 [19.28]
Max. Theoretical absorbed power	KW	14,9	19,1	24,8	29,3	30,1	34,2	35,3	42,6
Theoretical absorbed torque at 100 bar [1 450 PSI]	N.m [in.lbf]	11,3 [100]	14,5 [128]	18,8 [166]	22,2 [196]	22,8 [202]	26,0 [230]	28,4 [251]	32,3 [286]
Moment of inertia	kg.m ² [slug.ft ²]	0,0014 [0.0010]							
Internal charge pump	cm ³ /rev [in ³ /rev]	4,9 [0.30]; 7,0 [0.43] or 9,0 [0.55]							
Charge relief valve setting	bar [PSI]	From 6 [87] to 30 [435]*							
High pressure relief setting	bar [PSI]	Max. 350 [5 076]							Max. 300 [4 351]
Mounting flange		SAE A, SAE B							
Mass	kg [lb]	16,3 [35.9] with control M, N, S 18,8 [41.4] with controls A, B, C, D, P, Q, T							
Noise level	dBA	< 75							

* 30 bar [435 PSI] only at maximum revolutions.



Main dimensions

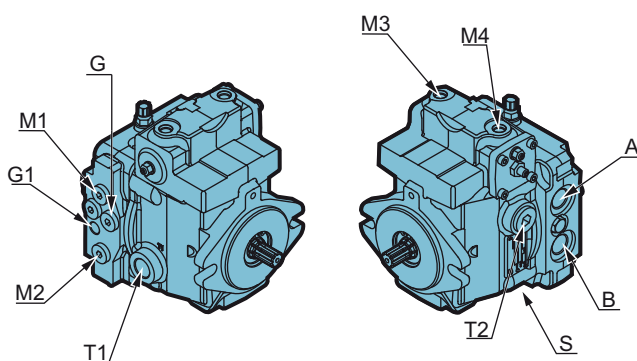
PM10 with hydraulic servo control and without auxiliary mounting pad.



	SAE A	SAE B
A	$\bar{\varnothing} 82,55^{0}_{-0,054}$ [dia. $3.25^{0}_{-0,002}$]	$\bar{\varnothing} 101,6^{0}_{-0,05}$ [dia. $4.00^{0}_{-0,002}$]
B	106 [4.17]	146 [5.75]
C	130 [5.12]	174 [6.85]
D	$\bar{\varnothing} 11,6$ [dia. 0.46]	$\bar{\varnothing} 14,3$ [dia. 0.56]
E	6,4 [0.25]	9,7 [0.38]

See from page 30 to page 41
for control dimensions.

Port characteristics



Port	Function	ISO 1179-1 (standard)	ISO 11926-1 (option FU)
A-B	Services	1/2" GAS	3/4-16 UNF-2B
G	Auxiliary	1/4" GAS	7/16-20 UNF-2B
M1/M2	Gauge	1/4" GAS	7/16-20 UNF-2B
M3/M4	Servo control pilot	1/8" GAS	7/16-20 UNF-2B
S	Suction	3/4" GAS	1-1/16-12 UNF-2B
T1	Drain	1/2" GAS	7/8-14 UNF 2B
T2	Drain	1/2" GAS	7/8-14 UNF-2B
G1	Auxiliary	1/4" GAS	1/4" GAS





OPERATING PARAMETERS

Operating parameters

			PM10-07	PM10-09	PM10-11	PM10-13	PM10-14	PM10-16	PM10-18	PM10-21
Speed ratings	Minimum						700			
	Max. without load	rpm					3 900 *			
	Max. with load						3 600			
System pressure	Rated						210 [3 045]			
	Maximum	bar [PSI]				350 [5 076]				300 [4 351]
	Minimum low loop					6 [87]				
Inlet pressure	Mini continuous	bar abs. [PSI abs.]				0,8 [11.6]				
	Mini (cold start)					0,5 [7.2]				
Case pressure	Continuous					1,5 [21.7]				
	Maximum (cold start)	bar [PSI]				2,5 [36.2]				
Charge pressure	Standard for M / N controls					10 [145]				
	Standard version	bar [PSI]				20 [290]				
	Max. charge pressure					30 [435] (only at maximum revolutions)				
Servo case pressure	Maximum	bar [PSI]				30 [435]				

* D control: max. speed limit is 3 600 rpm, with or without load.

Charge pressure

A charge flow is required to maintain a positive pressure in the low pressure loop of a closed loop hydrostatic transmission. Charge pressure ensures proper lubrication and rotating group operation. It is recommended to maintain the charge pressure at a minimum of 6 bar [87 psi] above case pressure. For more details, refer to charge pump paragraph, page 18.

Case pressure

Case pressure must be maintained within the limits shown in the table "Operating parameters". Ensure housing is always filled with hydraulic fluid and especially during start-up of the machine.

Pressure ratings

Maximum peak pressure

It is the maximum allowable pressure. It is equivalent to the maximum setting of the maximum high pressure relief valve. A self-propelled machine can reach the maximum peak pressure value no more than 1-2% of that work cycle.

Work cycle

A fundamental factor for ensuring correct hydrostatic transmission sizing is the machine work cycle (pressure-time ratio, seasonality, pressure vs. percentage of time at max. displacement, machine type). Part service life depends on the correct choice in relation to the work cycle.

Overloads

It is mandatory to protect parts against any possible overloads.

Speed ratings

The table "Operating parameters" gives minimum and maximum rated speeds. Note that all displacements might operate under different speed limits. Definitions of these speed limits appear below.

Maximum speed is the highest operating speed allowed. Over speeding reduces pump life time, can lead to loss of hydrostatic power and braking capacity. Never exceed the maximum speed limit under any operating conditions.

Nominal speed is the speed offering the maximal efficiency.

Model
CodeTechnical
specificationsOperating
ParametersSystem design
Parameters

Features

Controls

Options



Inlet pressure

Charge pump inlet pressure is key for acceptable pump life and performances. A continuous inlet pressure of not less than 0,8 bar abs. [11.6 PSI abs.] is recommended. An continuous inlet pressure less than 0.5 bar abs. [7.2 PSI abs.] indicates inadequate inlet design or a restricted filter. Pressures less than 0.5 bar abs. [7.2 PSI abs.] can happen at cold start, but should increase with oil temperature.

Theoretical output

Theoretical output flow is a function of pump displacement and speed. It is relevant to size the rest of the circuit. Theoretical flow does not take into account losses due to leakage or variations in displacement. Refer to performances, page 6, for volumetric and overall efficiencies at various operating speeds and pressures.

Poclain Hydraulics recommendations for fluid



Poclain hydraulics recommends the use of hydraulic fluids defined by the ISO 15380 and ISO 6743-4 standards. For temperate climates, the following types are recommended.

- HM 46 or HM 68 for fixed installations.
- HV 46 or HV 68 for mobile installations.
- HEES 46 for mobile installations.

These specifications correspond to category 91H of the CETOP standard, parts 1, 2 and 3 of the DIN 51524 standard, and grades VG32, VG 46 and VG68 of the ISO 6743-4 standards.



It is also possible to use ATF, HD, HFB, HFC or HFD type hydraulic fluid upon Poclain Hydraulics specific approval of the components' operating conditions.

Standardized designations for the fluids

- **HM** : Mineral fluids having specific antioxidant, anticorrosion and antiwear properties (HLP equivalent to DIN 51524 parts 1 and 2).
- **HV** : HM mineral fluids providing improved temperature and viscosity properties (DIN 51524 part 3).
- **HEES** : Biodegradable fluids based on organic esters.



It is also possible to use a fluid that meets the biodegradability criteria and is compatible in the event of accidental food contact. The BIOHYDRAN FG 46 fluid designed by the company Total has undergone testing of its properties and performance on our test benches. Since this type of fluid has not yet been categorized, it is the responsibility of machine manufacturers to validate its compatibility with all of the components used in order to guarantee that the intended functions will be fulfilled and this for the desired life time of all equipment items.



For biodegradable fluids, consult your Poclain Hydraulics' application engineer



During operation, the temperature of the oil must be between 0°C [32°F] and 80°C [176°F]; the minimum and maximum temperatures may be exceeded momentarily by ± 20°C [+ 68°F / - 4°F] for a duration of less than 30 minutes.

For all applications outside these limits, please consult with your Poclain Hydraulics' application engineer.

Pump storage



If the pump stays on stock for more than 6 months, a status verification must be performed before you install it on a machine. Pay attention to sealing condition, rust presence and free rotation of shaft.

Pump installation



You are strongly advised to follow installation instructions specified in Installation guide No. B61352L.



Fluid and filtration

The contaminating particles suspended in the hydraulic fluid cause the hydraulic mechanisms moving part wear. On hydraulic pumps, these parts operate with very small dimensional tolerances. In order to reach the part life, it is recommended to use a filter that maintains the hydraulic fluid contamination class at a max. of:

6 according to NAS 1638
17/15/12 according to ISO 4406:2021

According to the type of application decided for the pump, it is necessary to use filtration elements with a filtration ratio of:

$\beta_{20 \text{ to } 30} \geq 100$

Making sure that this ratio does not worsen together with the increasing of the filter cartridge differential pressure.

If these values cannot be observed, the component life will consequently be reduced and it is recommended to contact the Poclain Hydraulics Customer Service.

Filters on charge circuit

Filters on the charge circuit (F0-F2) are designed without by-pass. The max. pressure drop on the filtration part must not exceed 2 bar [29 PSI] (3 bar [43.5 PSI] in case of cold starting) at pump full rating. To monitor the pressure drop, it is recommended to use the clogging indicator on the filtration element (F2 option). Contact your Poclain Hydraulics Application engineer, each time the pump is not charged by its internal charge pump.

Filters on charge circuit are mounted on the pump special support.

Filters assembling

The suction filter is mounted on the suction line. Check that the pressure before the charge pump is 0.8 bar abs. [11.6 PSI abs.], measured on the pump suction port (0.5 bar [7.2 PSI] for cold starting).

Viscosity range

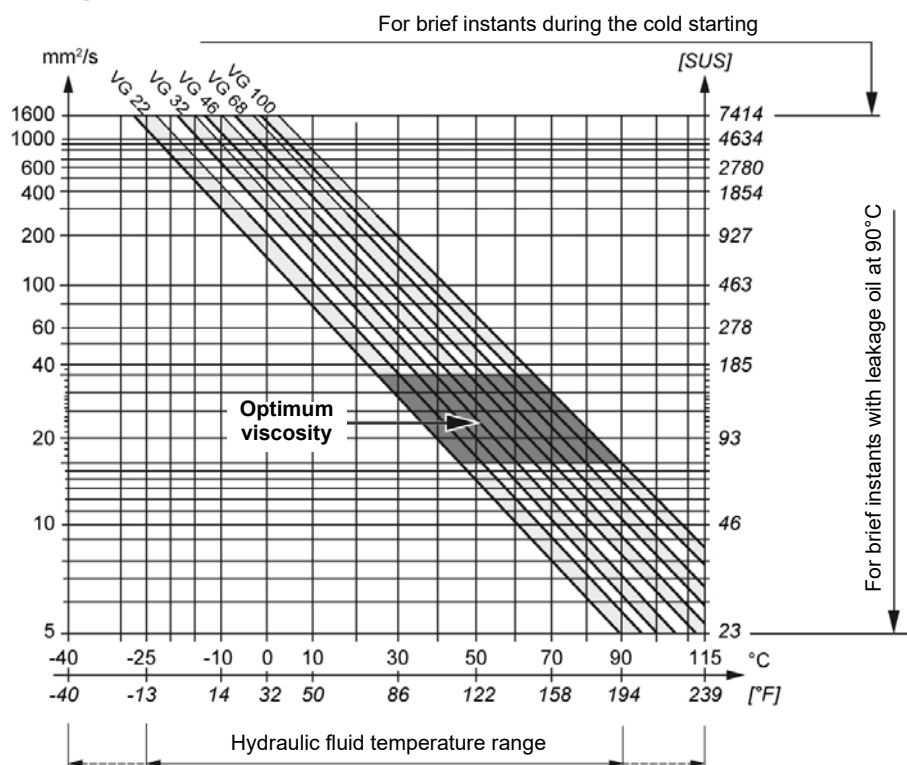
For both max. efficiency and life of the unit, the operative viscosity should be chosen within the optimum range of:

$\sqrt{\text{opt}}$ = optimum operating viscosity from 16 to 36 mm²/s [from 74.1 to 166.8 SUS] referred to the closed loop temperature.

Working conditions: the following limits of viscosity apply

$\sqrt{\text{min}}$ = 5 mm²/s [23 SUS] short-duration at a max. permissible leakage oil temperature of 90° C [194°F]

$\sqrt{\text{max}}$ = 1000 mm²/s [4 634 SUS] short-duration, on cold start.



Ensure fluid temperature and viscosity limits are concurrently satisfied.



SYSTEM DESIGN PARAMETERS



Consult your Poclain Hydraulics application engineer to validate your design parameters before using the pump in your application.

Sizing equations

The following equations are helpful when sizing hydraulic pumps. Generally, the sizing process is initiated by an evaluation of the machine system to determine the required motor speed and torque to perform the necessary work function. First, the motor is sized to transmit the maximum required torque. The pump is then selected as a flow source to achieve the maximum motor speed.

SI units	Output flow Q	$= \frac{V_g \cdot n \cdot \eta_v}{1000}$	(l/min)
	Input torque M	$= \frac{V_g \cdot \Delta p}{20 \cdot \pi \cdot \eta_m}$	(N.m)
	Input power P	$= \frac{M \cdot n \cdot \pi}{30\,000} = \frac{Q \cdot \Delta p}{600 \cdot \eta_t}$	(kW)
US units	Output flow Q	$= \frac{V_g \cdot n \cdot \eta_v}{231}$	[GPM]
	Input torque M	$= \frac{V_g \cdot \Delta p}{2 \cdot \pi \cdot \eta_m}$	[lbf.in]
	Input power P	$= \frac{M \cdot n \cdot \pi}{198\,000} = \frac{Q \cdot \Delta p}{1714 \cdot \eta_t}$	[hp]

V_g = Displacement per revolution cm^3/tr [in^3/rev]
 Δp = $p_o - p_i$ (system pressure) bar [PSI]
 n = Speed min^{-1} [rpm]
 η_v = Volumetric efficiency
 η_m = Mechanical efficiency
 η_t = Overall efficiency = $\eta_v \times \eta_m$

Redundant braking system requirement



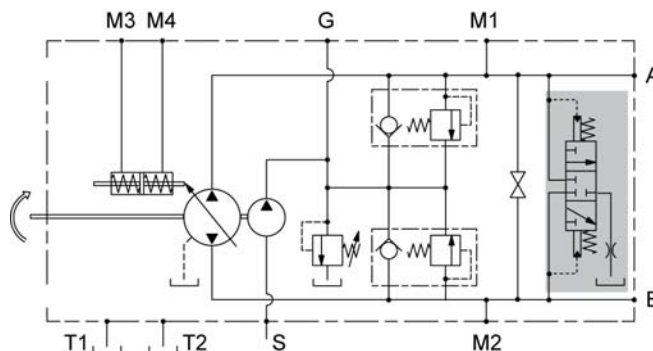
Unintended vehicle or machine movement hazard.

The loss of hydrostatic drive line power, in any mode of operation (forward, neutral, or reverse) may cause the system to lose hydrostatic braking capacity. You must provide a braking system, redundant to the hydrostatic transmission, sufficient to stop and hold the vehicle or machine in the event of hydrostatic drive power loss.

Loop flushing

Closed circuit may require a flushing valve to meet temperature and cleanliness requirements. A flushing valve takes a part of hot fluid flow from the low pressure loop of the system loop for cooling and filtering. Make sure that the charge pump provides adequate flow for the flushing valve flushing and the flushing valve does not cause charge pressure to drop below recommended limits.

See option VS for more information





Reservoir

The reservoir provides clean fluid, dissipates heat, and removes entrained air from the hydraulic fluid. It allows for fluid volume changes associated with fluid expansion and cylinder differential volumes. Minimum reservoir capacity depends on the volume needed to perform these functions. Typically, a capacity of one half the charge pump flow (per minute) is satisfactory for a closed reservoir. Open circuit systems sharing a common reservoir require greater fluid capacity.

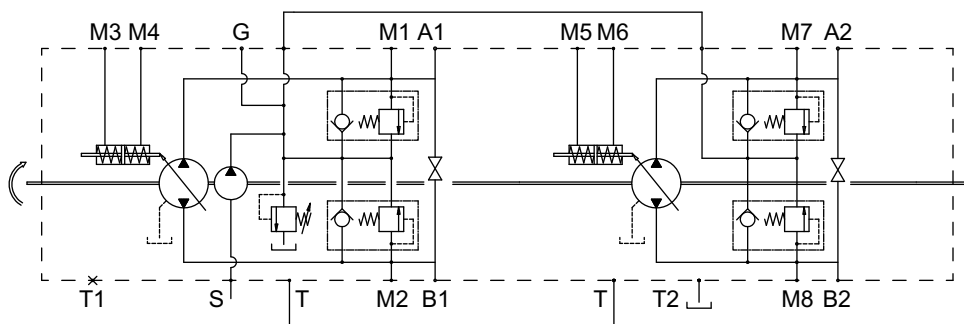
Locate the reservoir outlet (suction line) near the bottom, allowing clearance for settling foreign particles. Use a 100 - 125 µm screen covering the outlet port.

Place the reservoir inlet (return lines) below the lowest expected fluid level, as far away from the outlet as possible.

Use a baffle (or baffles) between the reservoir inlet and outlet ports to promote de-aeration and reduce fluid surging.

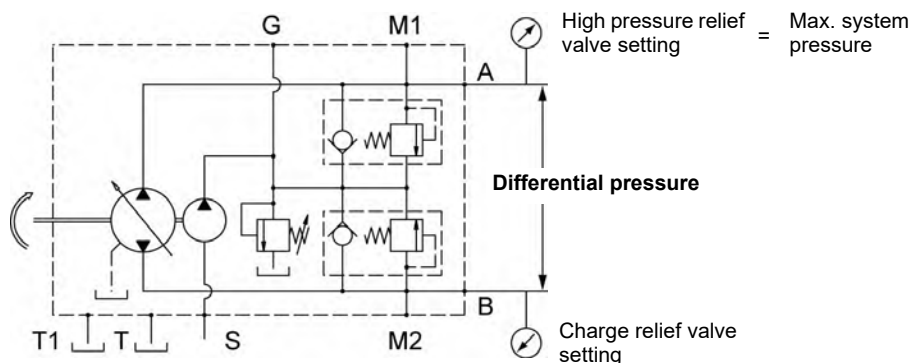
Case drain usage for tandem pump

On tandem pumps, and to ensure lubrication of both pumps, excess flow from the first pump charge relief valve must be routed into the housing of the second pump.



Differential pressure

The differential pressure is the High pressure relief valve setting minus Charge relief valve setting.





Bearing life and external shaft loading

Bearing life:

Bearing life is a function of speed, pressure, swashplate angle and external loads. Oil type and viscosity impact bearing life.

	PM10-07	PM10-09	PM10-11	PM10-13	PM10-14	PM10-16	PM10-18	PM10-21
Bearing life (B ₁₀ hours)	76 105	36 062	16 294	9912	9 204	8053	4 743	3 178

Normal bearing life in B₁₀ hours is shown in the table above. Figures have been calculated under the following operating conditions :

A continuous differential pressure of 120 bar [1740 PSI], 1800 rpm shaft speed, 20 bar [290 PSI] charge pressure, maximum displacement, without any external shaft side load. The data is based on a 50% forward, 50% reverse duty cycle, and standard charge pump size.

Shaft Loads

PM10 pumps are designed with bearings that can accept external radial and thrust loads. The external radial shaft load limits depend on the load position, orientation, and operating conditions of the unit.

The maximum permissible radial load (Re), is based on the maximum external moment (Me), and the distance (L) from the mounting flange to the load. It may be determined using the table and formula below. Thrust (axial) load limits are also shown.

$$Re = Me / L$$

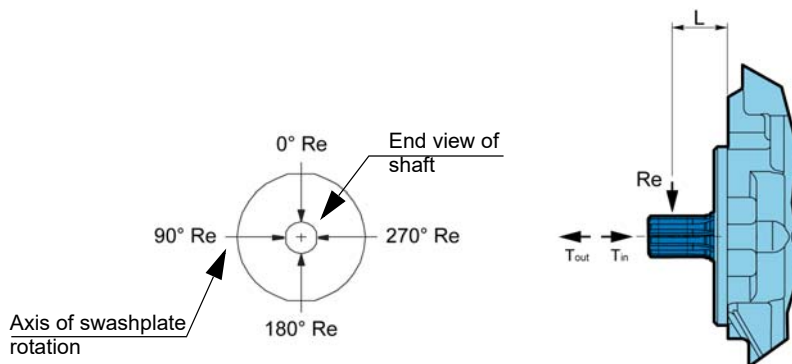
All external shaft loads affect bearing life. In applications with external shaft loads, minimize the impact by positioning the load at 90° or 270° as shown in the figure.

Contact your Poclain Hydraulics representative for an evaluation of unit bearing life if:

- Continuously applied external loads exceed 25 % of the maximum allowable radial load Re.
- The pump swashplate is positioned on one side of center all or most of the time.
- The unit bearing life (B₁₀) is critical.

	PM10-07	PM10-09	PM10-11	PM10-13	PM10-14	PM10-16	PM10-18	PM10-21
External moment (Me) N.m [in.lbf]	63 [558]	59 [522]	52 [460]	49 [438]	46 [407]	40 [383]	38 [336]	32 [283]
at 120 bar [1740 PSI]								

Radial and thrust load position



For an accurate calculation, consult your Poclain Hydraulics application engineer and use new AXEL program.



Hydraulic unit life

Hydraulic unit life is the life expectancy of the hydraulic components. It depends on speed and system pressure even if , system pressure is the dominant operating variable. High pressure, generated by high load, reduces hydraulic unit life.

Design the hydraulic system according to the expected machine duty cycle. Take in consideration the expected percentages of time at various loads and speeds. Ask your Poclain Hydraulics representative to calculate an appropriate pressure based your hydraulic system design. If duty cycle data is not available, input power and pump displacement are used to calculate system pressure.

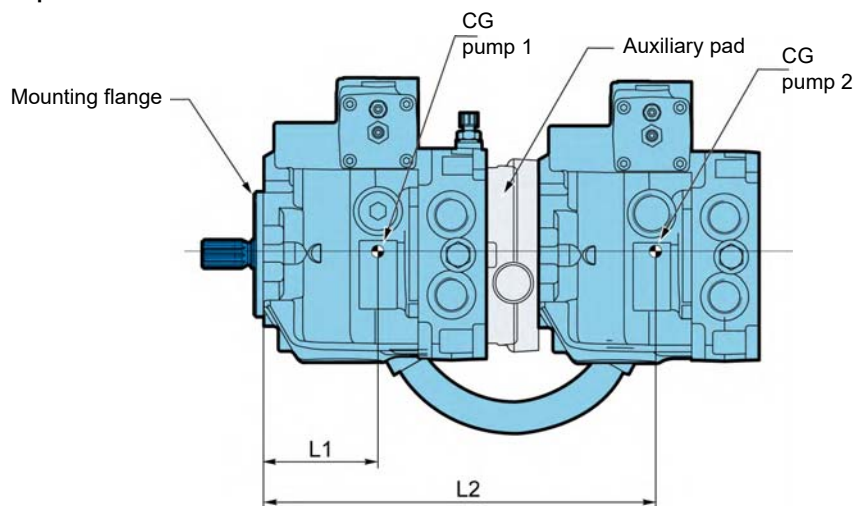
All pressure limits are differential pressures (referenced to charge pressure) , taking a normal charge pressure in consideration.

PM10 pumps will meet satisfactory life expectancy if applied within the parameters specified in this technical documentation. For more detailed information on hydraulic unit life see Operating Parameters in page 9.

Mounting flange loads

Adding tandem mounted pumps, and/or tandem auxiliary pump(s), subjecting pumps to shock loads may generate excessive loads on the front mounting flange. The overhung load moment for multiple pump mounting can be estimated as shown in the figure below.

Overhung load example



Estimating overhung load moments

W = Weight of pump (kg)

L = Distance from mounting flange to pump center of gravity (CG)

$$M_R = G_R (W_1 L_1 + W_2 L_2 + \dots + W_n L_n)$$

$$M_S = G_S (W_1 L_1 + W_2 L_2 + \dots + W_n L_n)$$

Where:

M_R = Rated load moment (N.m)

M_S = Shock load moment (N.m)

G_R^* = Rated (vibratory) acceleration (G's) (m/sec²)

G_S^* = Maximum shock acceleration (G's) (m/sec²)

*Calculations will be carried out by multiplying the gravity ($g = 9.81 \text{ m/sec}^2$) with a given factor. This factor depends on the application.

Allowable overhung load moment are shown in the above table. Exceeding these values requires additional pump support.



For an accurate calculation, consult your Poclain Hydraulics application engineer.



FEATURES

High pressure relief valve

The High pressure relief valves maintain circuit pressure in the proper range. The check valves allow charge flow to replenish the low pressure loop of the circuit. The high pressure relief valves ensure a high pressure protection of the high pressure loop of the circuit.

High pressure relief valves are available in a wide range of settings.

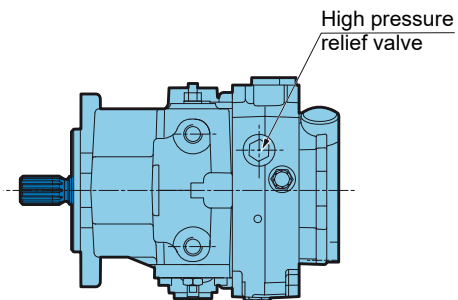
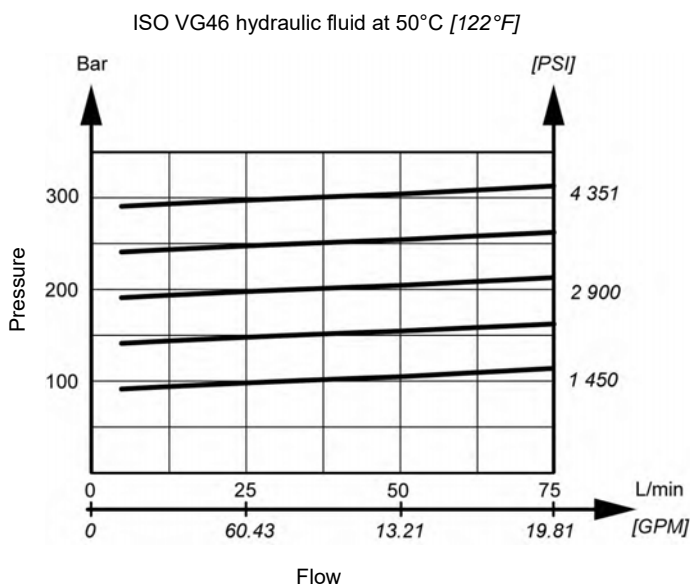
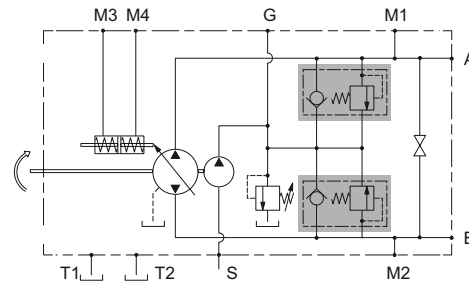
When high pressure relief valves are not desired, pumps is equipped with charge circuit check valves only. The High pressure relief valve are not adjustable. To change setting is necessary to change the whole valve.



High pressure relief valves are intended for transient overpressure protection and are not intended for continuous pressure control. Flow over relief valves for extended periods of time may result in severe heat build up. High flows over relief valves may result in pressure levels exceeding the nominal valve setting and potential damage to system components.



High pressure relief valve	Available setting bar [PSI]	
Without	-	00
	100 [1 450]	10
	150 [2 175]	15
With	200 [2 900]	20
	250 [3 625]	25
	300 [4 351]	30
	350 [5 076]	35



The high pressure relief valve setting is not the differential pressure between A and B ports (see page 13).



Charge relief valve

The charge pressure relief valve provides a relief outlet for charge circuit. This valve is used to set the charge pressure of the circuit. Flow through the valve is ported to case.

The nominal charge relief setting is referenced to case pressure.



Incorrect charge pressure settings may result in the inability to build required system pressure and/or inadequate loop flushing flows. Ensure correct charge pressure under all conditions of operation to maintain pump control performance.

P

M

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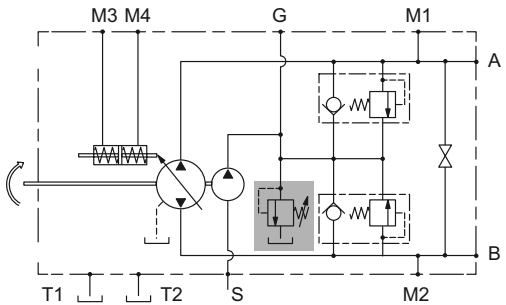
11

12

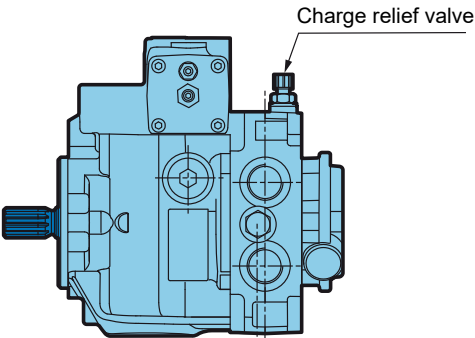
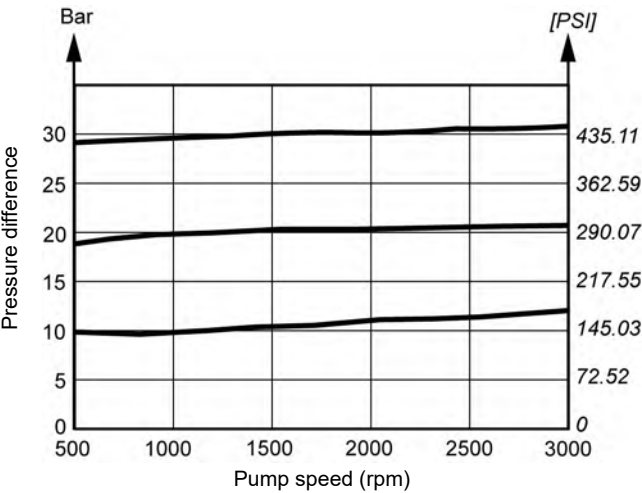
Charge relief valve

Available setting
bar [PSI]

Without	-	00
With	For M and N controls	10 [145]
	For servo controls	20 [290]



ISO VG46 hydraulic fluid at 50°C [122°F]



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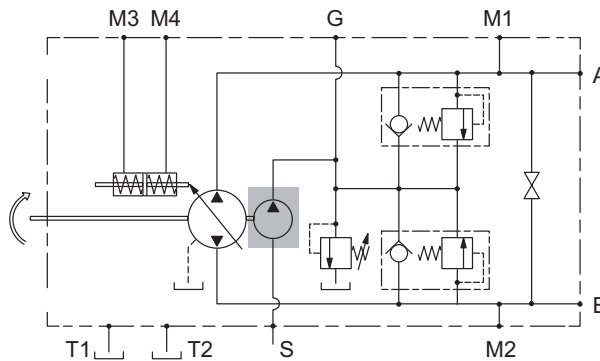


Charge pump

Charge flow is required on all PM10 pumps used in closed circuit installations. The charge pump provides flow to make up internal leakage, maintain a positive pressure in the main circuit, provide flow for cooling and filtration, replace any leakage losses from external valving or auxiliary systems, and to provide flow and pressure for the control system.

Many factors influence the charge flow requirements. These factors include system pressure, pump speed, pump swashplate angle, type of fluid, temperature, size of heat exchanger, length and size of hydraulic lines, control response characteristics, auxiliary flow requirements, hydrostatic motor type, etc.

Unusual application conditions may require a more detailed review of charge pump sizing. Charge pressure must be maintained at a specified level under all operating conditions to prevent damage to the transmission. Poclain Hydraulics recommends testing under actual operating conditions to verify this.



Charge pump sizing/selection

In most applications, a general guideline is that the charge pump displacement should be at least 20% of the main pump displacement.

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
---	---	---	---	---	---	---	---	---	---	---	---	---	----	----	----

Charge pump		Displacement cm ³ /rev [in ³ /rev]	Rated speed (rpm)	
Without		-	-	00
With	For auxiliary mounting pad F and G	4,9 [0.30]	3900	05
	For auxiliary mounting pad A	7,0 [0.43]	3900	07
	For tandem pump with auxiliary mounting pad A and single pump	9,0 [0.55]	3900	08

Pump without internal charge pump is also available. In this case an external flow must provide charge pressure and charge flow in order to assure the requested pump performance.



Contact your Poclain Hydraulics application engineer for more information.



Pump version without internal charge pump is available. In this case an external flow must provide charge pressure and charge flow in order to assure the requested pump performance.



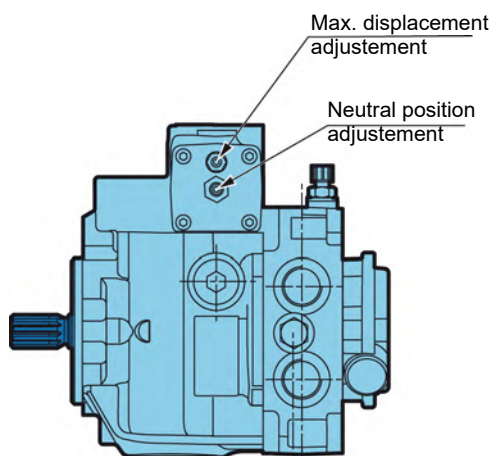
Displacement limiters

PM10 are designed with mechanical displacement (stroke) limiters. You can limit maximum displacement of the pump to a certain per cent of its maximum displacement to near zero in both direction.

The displacement limiters are located on the both sides of the servo piston and are adjustable by screw.

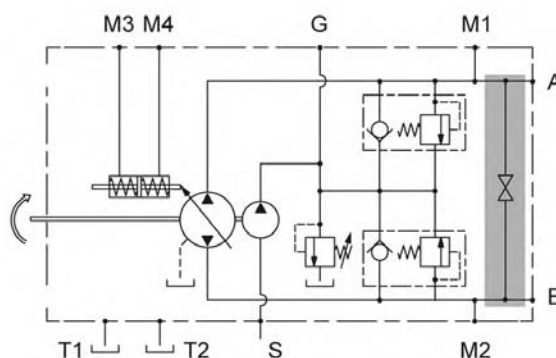
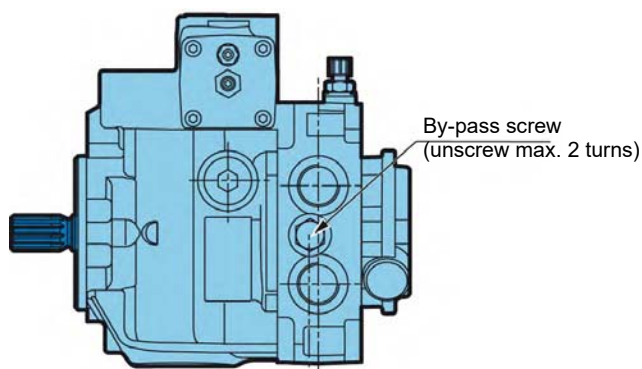


Take care in adjusting displacement limiters to avoid an undesirable condition of output flow or speed. Retorque the sealing lock nut after every adjustment to prevent an unexpected change in output conditions and to prevent external leakage during pump operation.



By-pass

PM10 features a by-pass function. By-passing Port A and Port B is achieved by unscrewing a screw located on the cover. The by-pass connect the ports A-B and must be use only in emergency case and only for short movement.



To avoid leakage, do not exceed two turns of the screw.



By-pass valve is intended for moving a machine for very short distances at very slow speeds. It is not intended as tow valve.

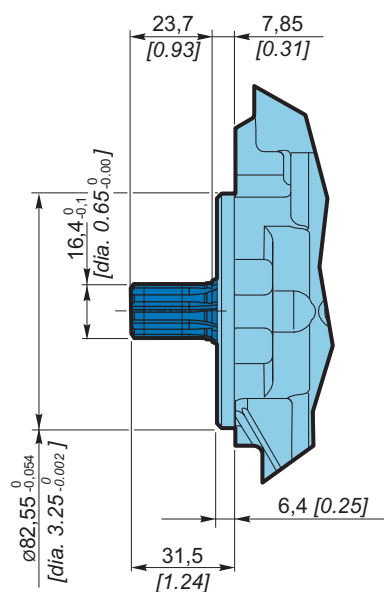


Mounting flange and shafts

SAE A - Splined shaft

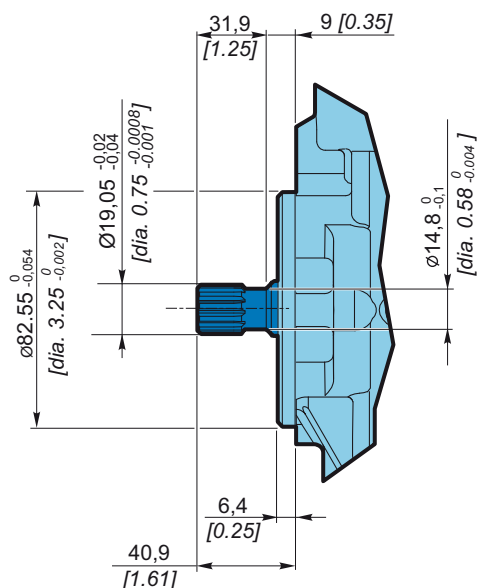
	1	2	3	4	5	6	7	8	9	10	11	12
P	M	1	0	A								

S1 9 teeth; Max torque: 80 N.m [708 in.lbf]



Splined ANSI B92.1a-1996
Pitch 16/32" DP
Pressure angle 30°
Tolerance class: 5

S2 11 teeth; Max torque: 140 N.m [1 239 in.lbf]



Splined ANSI B92.1a-1996
Pitch 16/32" DP
Pressure angle 30°
Tolerance class: 5



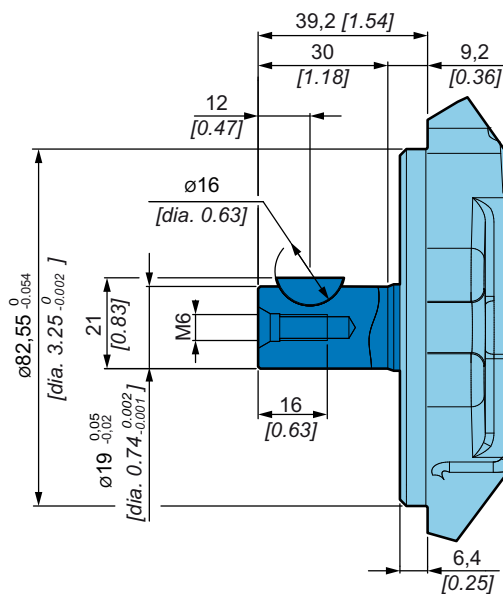
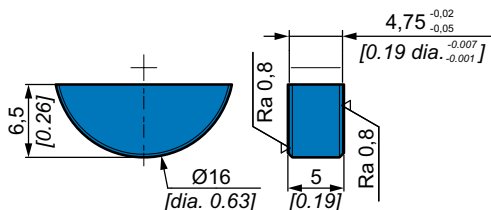
SAE A - Shaft with feather key

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
				A											

C2 Ø19 shaft; Max torque: 140 N.m [1 239 in.lbf]

Dimensions of key:

Definition of edges according to ISO 13 715



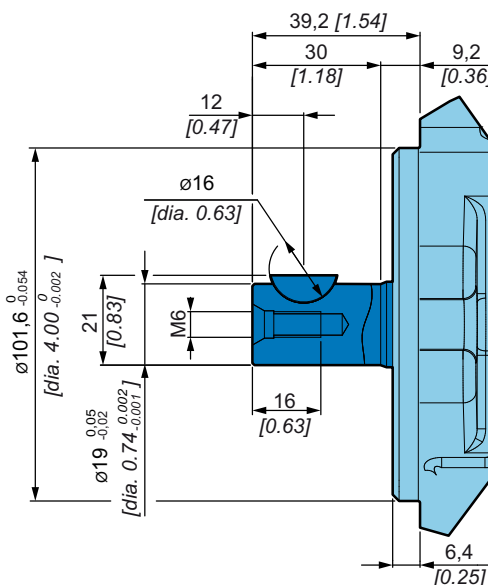
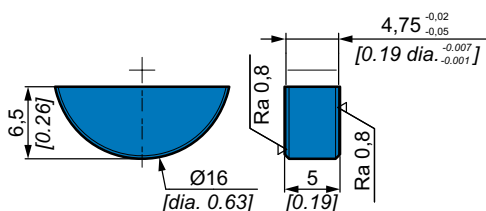
SAE B - Shaft with feather key

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
				B											

C2 Ø19 shaft; Max torque: 140 N.m [1 239 in.lbf]

Dimensions of key:

Definition of edges according to ISO 13 715

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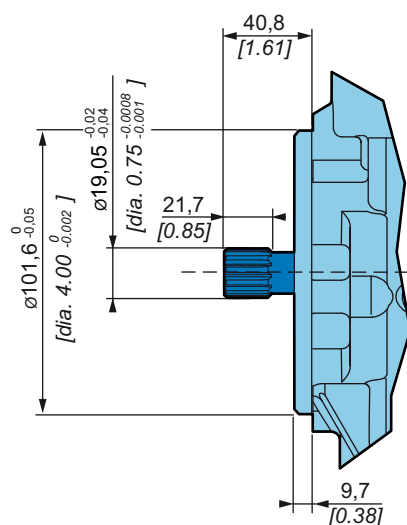
Options



SAE B - Splined shaft

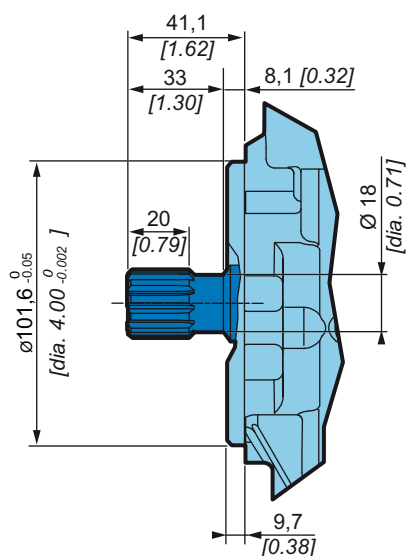
P	M	1	0	B	1	2	3	4	5	6	7	8	9	10	11	12
---	---	---	---	---	---	---	---	---	---	---	---	---	---	----	----	----

S2 11 teeth; Max torque: 140 N.m [1 239 in.lbf]



Splined ANSI B92.1a-1996
Pitch 16/32" DP
Pressure angle 30°
Tolerance class: 5

S3 13 teeth; Max torque: 220 N.m [1 947 in.lbf]



Splined ANSI B92.1a-1996
Pitch 16/32" DP
Pressure angle 30°
Tolerance class: 5



Auxiliary mounting pad

SAE A flange

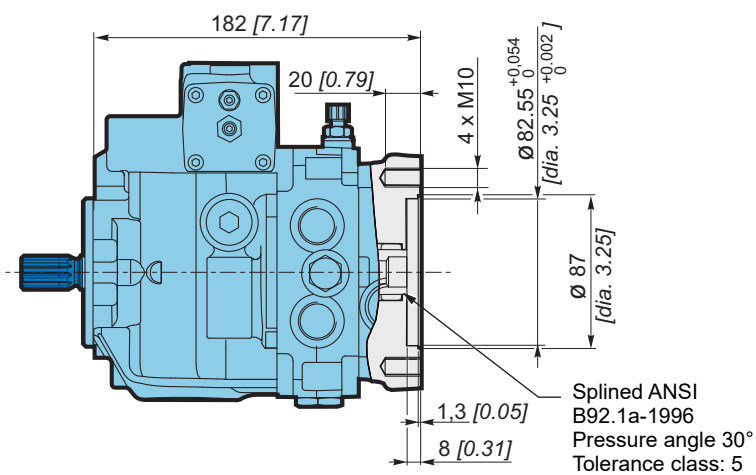
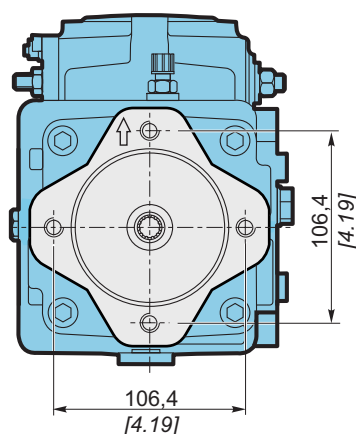


00 Without charge pump

07 With charge pump: 7,0 cm³/rev [0.43 in³/rev]

08 With charge pump: 9,0 cm³/rev [0.55 in³/rev]

Flange type	Number of teeth	Pitch	Max. torque N.m [in.lbf]	
SAE A	9	5/8" pitch 16/32" DP	80 [708]	A
	11	3/4" pitch 16/32" DP	125 [1 106]	E



Do not rotate the auxiliary mounting pad cover.

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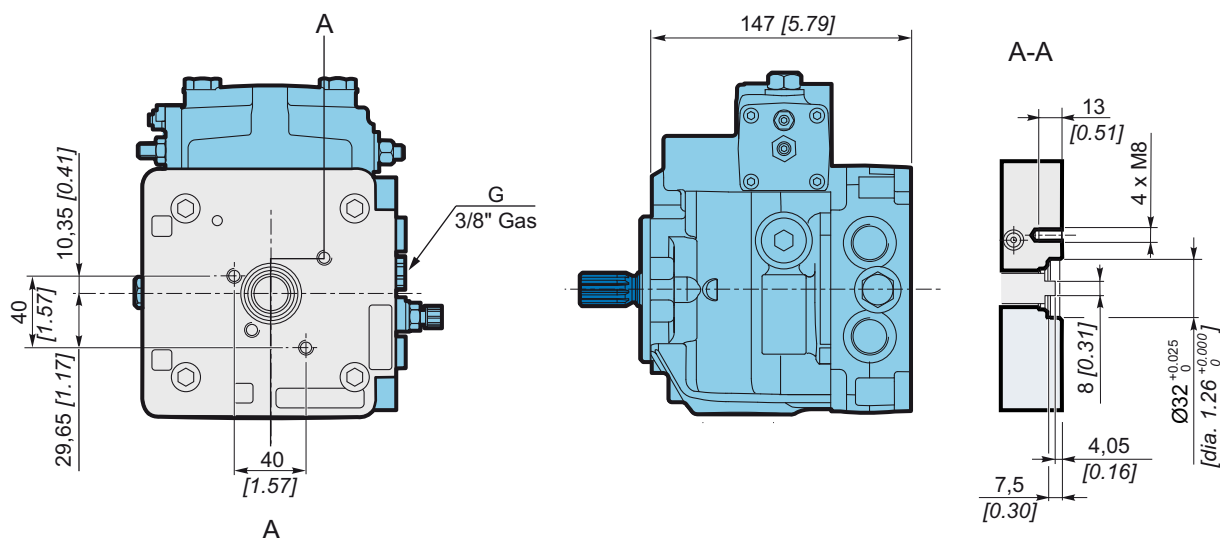


German group 1 flange

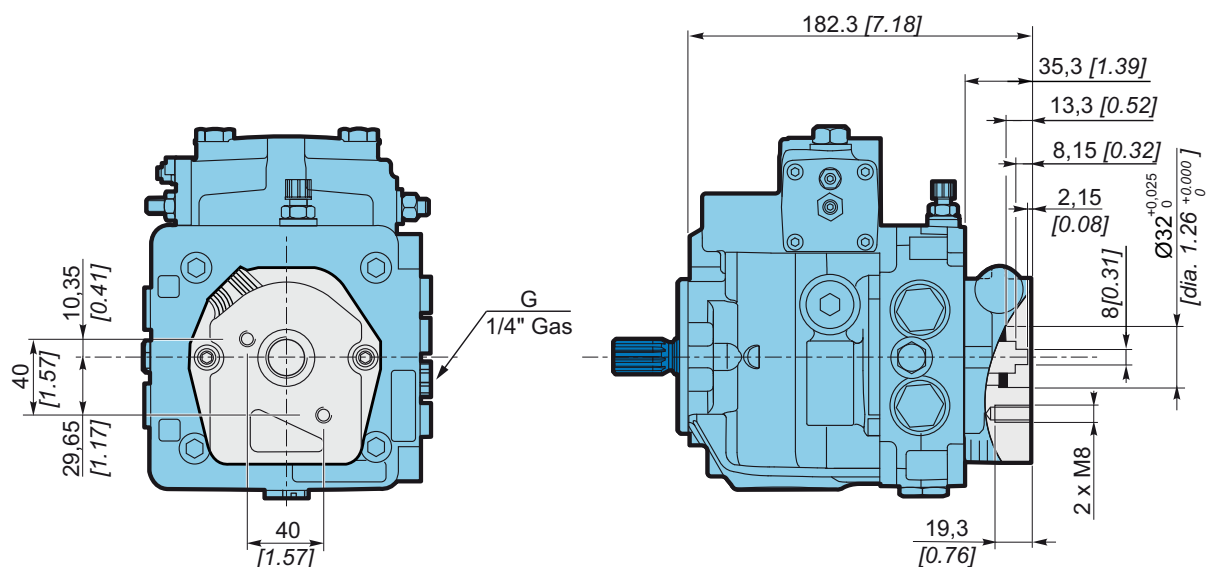
Max. torque: 48 N.m [425 in.lbf]

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
													F		

00 Without charge pump



05 With charge pump: 4,9 cm³/rev [0.30 in³/rev]



Do not rotate the auxiliary mounting pad cover.

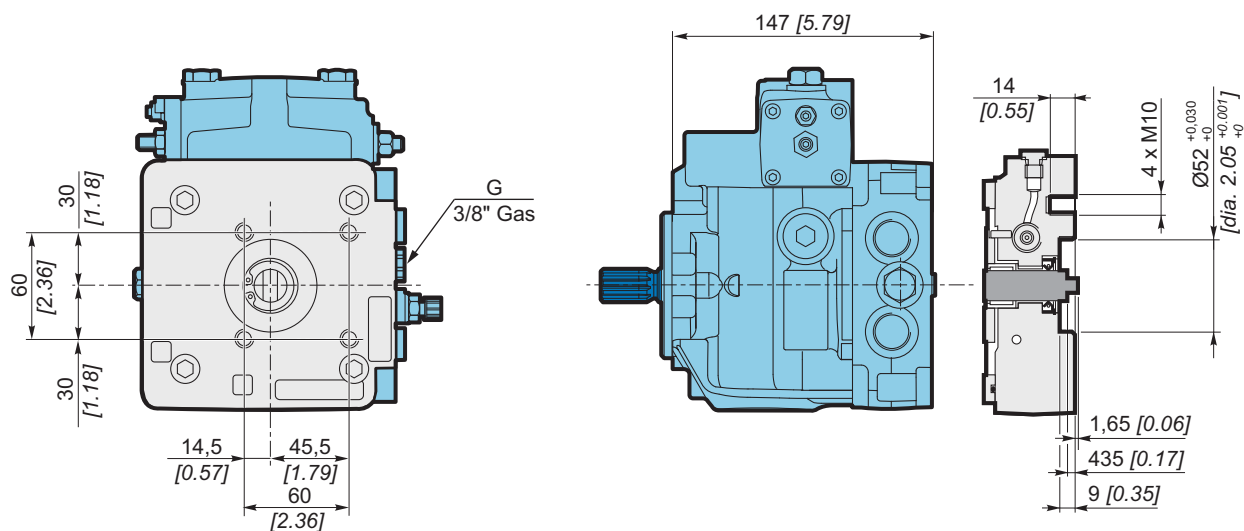
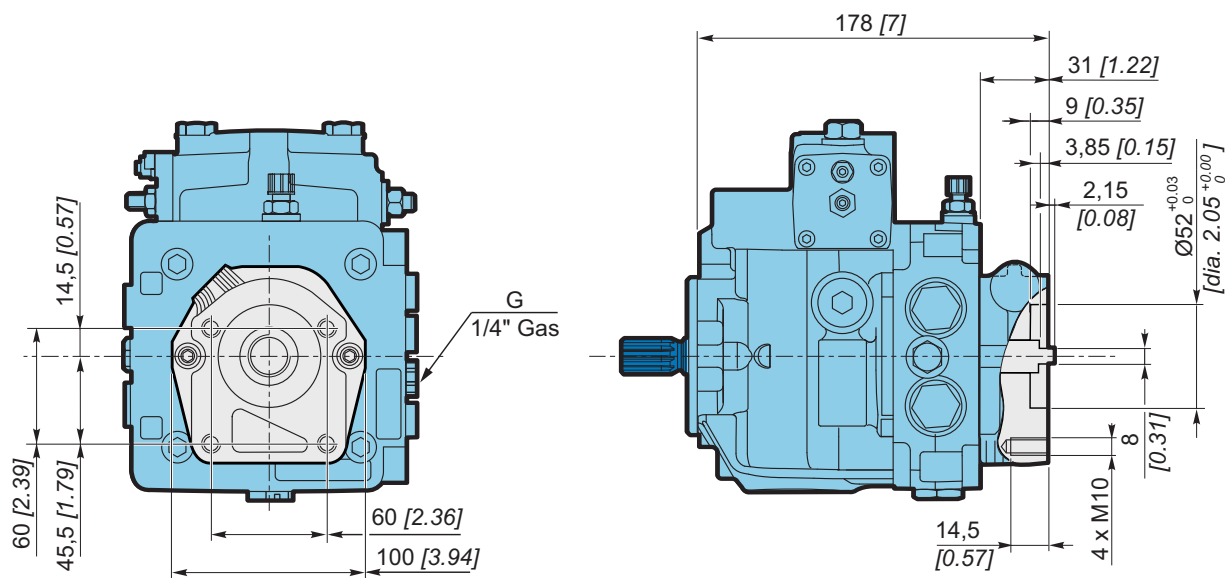


German group 2 flange

Max. torque: 70 N.m [620 in.lbf]

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
													G		

00 Without charge pump

05 With charge pump: 4,9 cm³/rev [0.30 in³/rev]

Do not rotate the auxiliary mounting pad cover.

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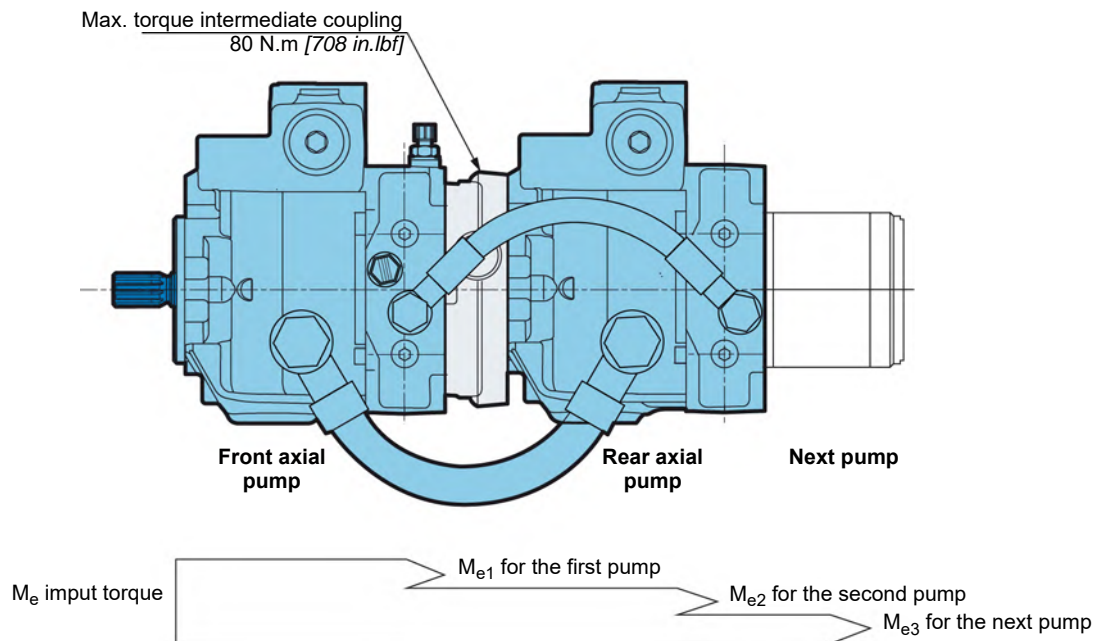
Features

Controls

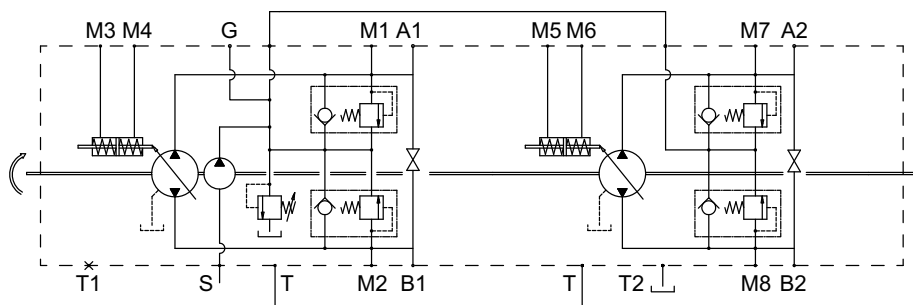
Options



Tandem pumps



**Torque required by gear pumps is additive.
Ensure requirements don't exceed shaft torque ratings.**



**Ports T and G of the first pump must be connected
with ports T and G of the second pump.**



Front axial pump	P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
Rear axial pump	P	M	1	0	A		S1									

Number of charge pump in the tandem	Axial pump	Mounting flange and shaft	Charge pump	Auxiliary mounting flange
1 charge pump*	Front	SAE A; 11 teeth	With*	07
		SAE B; 11 teeth		08
		SAE B; 13 teeth		
	Rear	SAE A; 9 teeth	Without	00
				Without auxiliary mounting pad
				German group 2
2 charge pumps	Front	SAE A; 11 teeth	With	07
		SAE B; 11 teeth		08
		SAE B; 13 teeth		
	Rear	SAE A; 9 teeth	With	05
				Without auxiliary mounting pad
				German group 2
				07
				08



* The charge pump in the front axial pump is mandatory.

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Gear pumps

P

M

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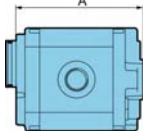
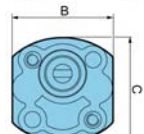
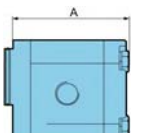
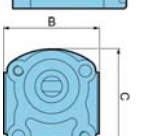
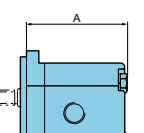
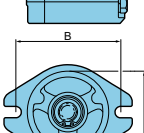
8

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12

		Auxiliary mounting pad									
		Gear pump									
		Displacement		Pressure			Dimension			Mass	Efficiency
		cm ³ /rev [cu.in/rev]	Continuous max. pressure	Max. intermittent pressure	Max. peak pressure	A	B	C	kg [lb]	%	
			bar [PSI]	bar [PSI]	bar [PSI]	mm [in]	mm [in]	mm [in]			
F	German group 1	01	1,6 [0.10]	210 [3 045]	240 [3 480]	260 [3 770]	76,4 [3.01]	67 [2.64]	70 [2.76]	0,95 [2.09]	95*
		02	2,0 [0.12]	210 [3 045]	240 [3 480]	260 [3 770]	77,9 [3.07]			0,97 [2.14]	
		03	3,2 [0.19]	200 [2 900]	240 [3 480]	250 [3 625]	82,6 [3.25]			1,04 [2.29]	
		04	4,2 [0.26]	180 [2 610]	210 [3 045]	230 [3 335]	86,5 [3.41]			1,10 [2.43]	
		05	5,0 [0.30]	180 [2 610]	210 [3 045]	230 [3 335]	89,6 [3.53]			1,14 [2.51]	
		06	6,3 [0.38]	170 [2 465]	190 [2 755]	210 [3 045]	94,7 [3.73]			1,22 [2.69]	
G	German group 2	04	4,5 [0.27]	250 [3 625]	270 [3 915]	290 [4 205]	90,3 [3.55]	88 [3.46]	100 [3.94]	2,30 [5.07]	95*
		06	6,0 [0.37]	250 [3 625]	270 [3 915]	290 [4 205]	93,6 [3.68]			2,45 [5.40]	
		08	8,5 [0.52]	250 [3 625]	270 [3 915]	290 [4 205]	97,8 [3.85]			2,60 [5.73]	
		11	11,0 [0.67]	250 [3 625]	270 [3 915]	290 [4 205]	101,9 [4.01]			2,70 [5.95]	
		14	14,5 [0.88]	250 [3 625]	270 [3 915]	290 [4 205]	106,9 [4.21]			2,80 [6.17]	
		17	17,0 [1.04]	230 [3 335]	240 [3 480]	250 [3 625]	111,1 [4.37]			2,95 [6.51]	
A	SAE A	04	4 [0.24]	250 [3 625]	270 [3 915]	290 [4 205]	93,0 [3.66]	106,4 [4.19]	82,5 [3.25]	2,30 [5.07]	95*
		06	6,0 [0.37]	250 [3 625]	270 [3 915]	290 [4 205]	96,3 [3.68]			2,45 [5.40]	
		08	8,5 [0.52]	250 [3 625]	270 [3 915]	290 [4 205]	100,5 [3.96]			2,60 [5.73]	
		11	11,0 [0.67]	250 [3 625]	270 [3 915]	290 [4 205]	104,6 [4.12]			2,70 [5.95]	
		14	14 [0.85]	250 [3 625]	270 [3 915]	290 [4 205]	109,6 [4.21]			2,80 [6.17]	
		17	16,5 [1.01]	230 [3 335]	240 [3 480]	250 [3 625]	113,8 [4.37]			2,95 [6.51]	
		20	19,5 [1.19]	210 [3 045]	220 [3 190]	230 [3 335]	118,8 [4.68]			3,10 [6.84]	

Gear pumps are always delivered flanged on the axial pump. They can not be sold alone.

* Value collected during the testing at 1500 rpm.

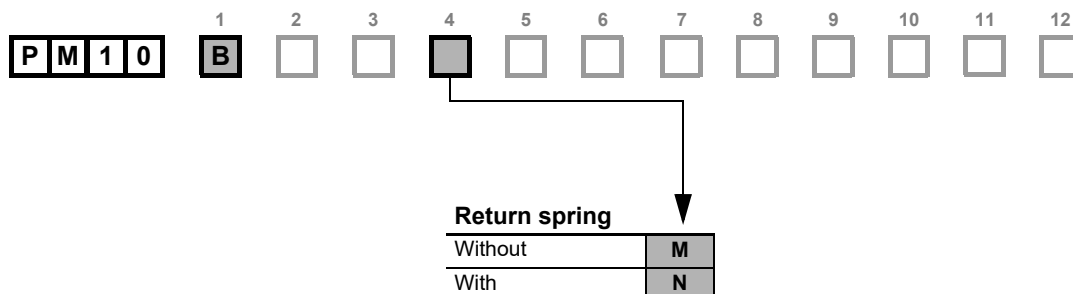


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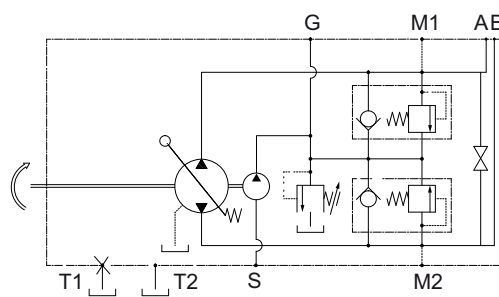


CONTROLS

Direct mechanical controls



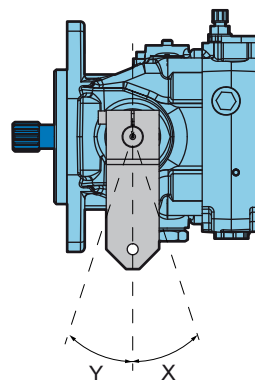
The variation in pump displacement is obtained by rotating the lever shaft in a clockwise or counter-clockwise direction.



Flow rate determination

Rotation	Pressure	Output	Input
Clockwise (R)	X	A	B
	Y	B	A
Counter clockwise (L)	X	B	A
	Y	A	B

Pump Max. displacement cm ³ /rev [in ³ /rev.]	Angle to reach max. displacement
7,08 [0.43]	11°
9,08 [0.55]	14°
11,83 [0.72]	18°
14,32 [0.87]	17°
17,85 [1.09]	18°
20,40 [1.24]	19°



The mechanical linkage built by the customer to stroke the pump should be able to return the pump to neutral in all conditions.



The spring return feature in the control unit is not a safety device.



P

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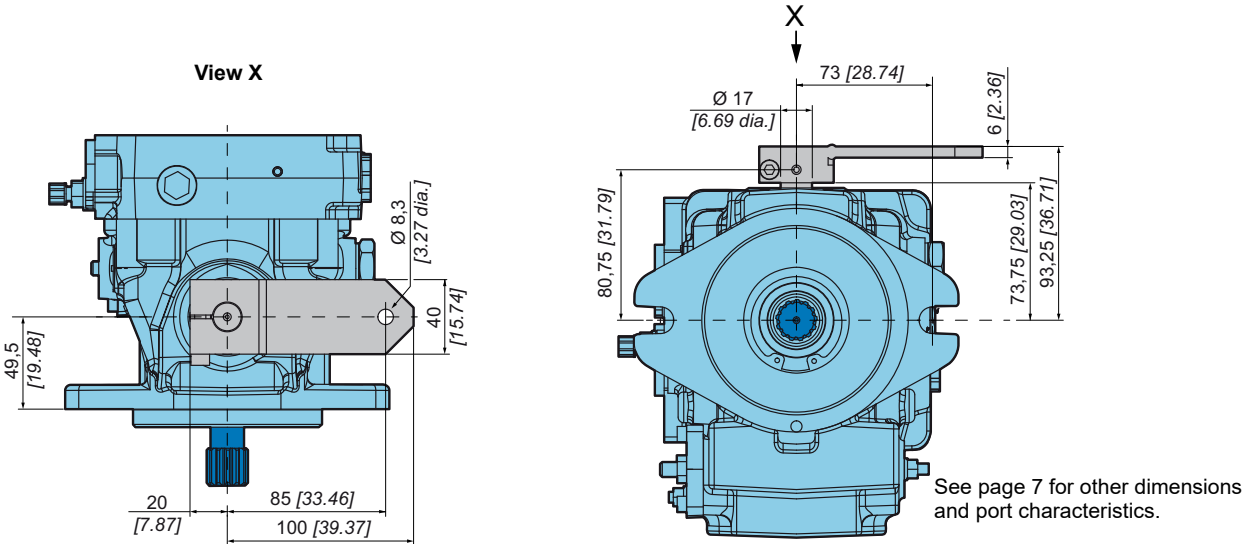
12

B

Control position

		M	Spring diameter	
			Ø2,8 mm [dia. 0.110]	Ø3,0 mm [dia. 0.118]
Control on the top	Without lever	MA0	NA028	NA030
	Left	MAL	NAL28	NAL30
	Right	MAR	NAR28	NAR30
	Without lever	MB0	NB028	NB030
	Left	MBL	NBL28	NBL30
	Right	MBR	NBR28	NBR30

Dimensions with controls M-N



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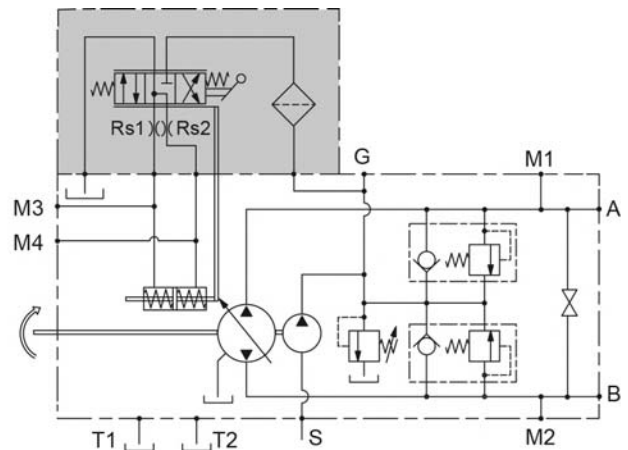
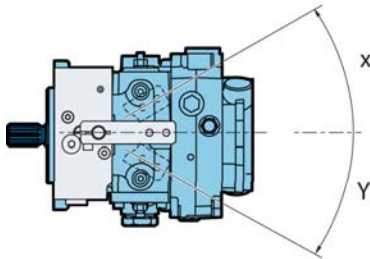
Mechanical servo control with feedback



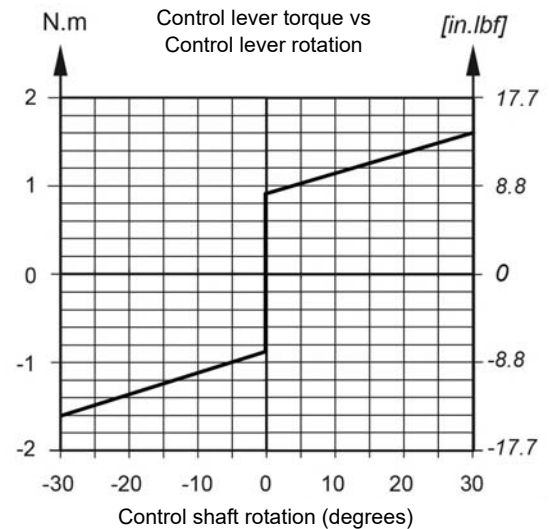
Control function	The variation in pump displacement is reached by control lever rotation to adjust hydraulic servo piston position. Control lever range is 34°. Movement of control lever is independent of the pressure and pump speed.
Control regulation	To avoid sudden accelerations and stoppages, two restrictors (Rs1 and Rs2) are inserted between servo control and hydraulic servo piston. They are used to regulate control shifting speed.
Feedback function	The feedback system between swash plate and hydraulic servo piston permit to maintain constant displacement of the pump if the pressure between pump and hydraulic motor changes. The feedback function is reached by a lever that connects the swashplate and the hydraulic servo piston.

Flow rate determination

Rotation	Control	Output	Input
Clockwise (R)	X	A	B
	Y	B	A
Counter clockwise (L)	X	B	A
	Y	A	B



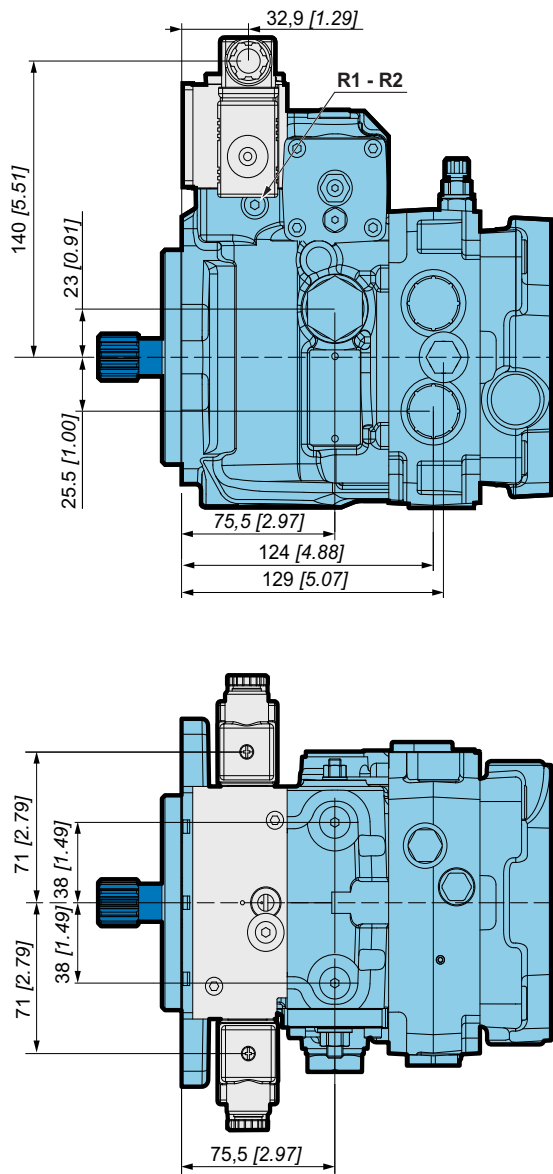
To prevent damage to the control A a positive mechanical stop must be provided for the control A linkage.



The spring return feature in the control unit is not a safety device.



Dimensions with control A



See page 7 for other dimensions and port characteristics.

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Hydraulic servo control



Control function	The variation in pump displacement is reached by pressure adjustment on the M3 and M4 servo control ports. These ports are controlled by hydraulic proportional joystick (containing pressure reduction valves). The joystick supply can be obtained by taking pressure from the auxiliary pump (R connection). Basic joystick can be provided upon request.
Control regulation	The servo control response time can be adjusted by two restrictors (Rs1 and Rs2) inserted on the joystick supply line.



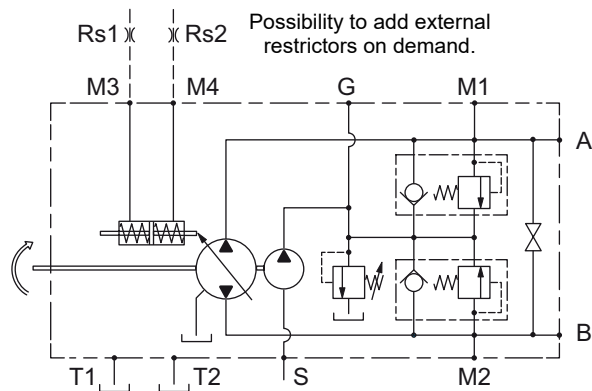
Other curves can be used in relation to valve plate timing. Contact your Poclain Hydraulics application engineer for further info.



For the selection of the regulation curve (with or without step) of the Joy-stick contact your Poclain Hydraulics application engineer.

Flow rate determination

Rotation	Pressure	Output	Input
Clockwise (R)	M3	B	A
	M4	A	B
Counter clockwise (L)	M3	A	B
	M4	B	A

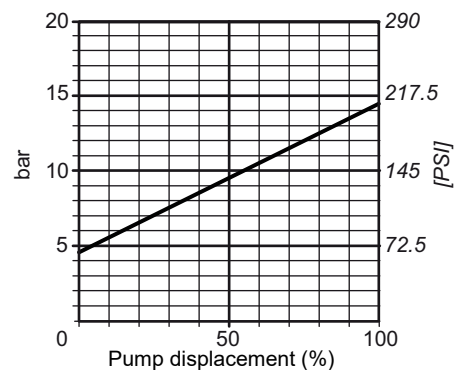


The spring return feature in the control unit is not a safety device.



The back pressure of the return line of the joystick and the drive line of the pump have an influence on Servo pressure vs Displacement values.

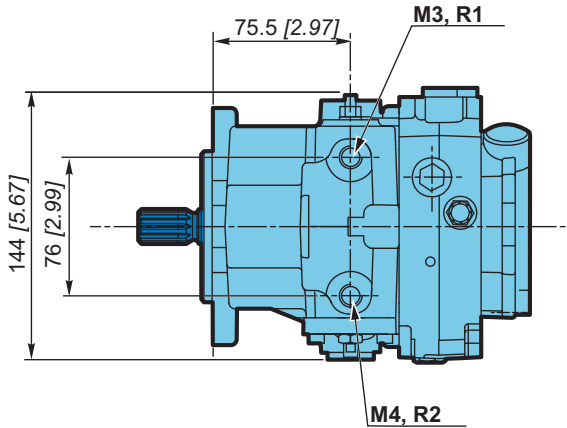
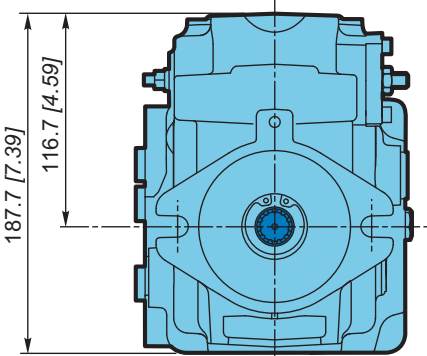
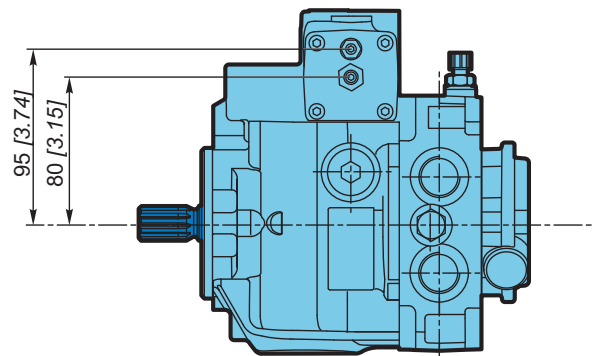
Servo pressure vs Displacement



Above graph is just an example that shows the relationship between servo pressure and displacement.



Dimensions with control S



See page 7 for other dimensions and port characteristics.

Model Code	Technical specifications	Operating Parameters	System design Parameters	Features	Controls	Options
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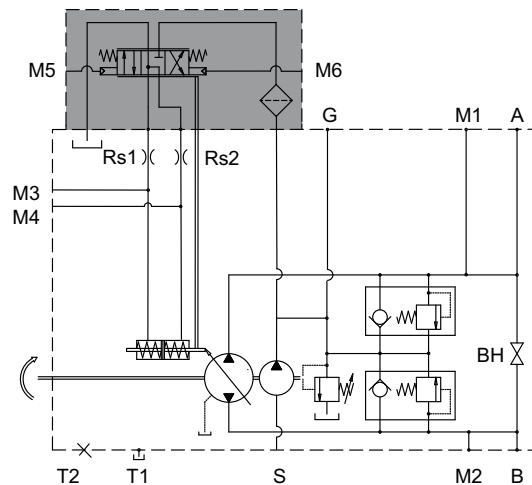
Hydraulic servo control with feedback



Control function	The variation in pump displacement is reached by pressure adjustment on the M5 and M6 feedback control ports. These ports are controlled by hydraulic proportional joystick (containing pressure reduction valves). The joystick supply can be obtained by taking pressure from the auxiliary pump (R connection). Basic joystick can be provided upon request
Control regulation	The servo control operation curve in both directions goes from 6 to 15 bar <i>[from 87 to 218 PSI]</i> . The adjustment curve of the hydraulic control system has to be wider, from 5 to 16 bar <i>[from 73 to 232 PSI]</i> .
Feedback function	The feedback system between swash plate and hydraulic servo piston permit to maintain constant displacement of the pump if the pressure between pump and hydraulic motor changes. The feedback function is reached by a lever that connects the swashplate and the hydraulic servo piston. To avoid sudden accelerations and stoppages, two restrictors (Rs1 and Rs2) are inserted between the servo control and the hydraulic servo piston.

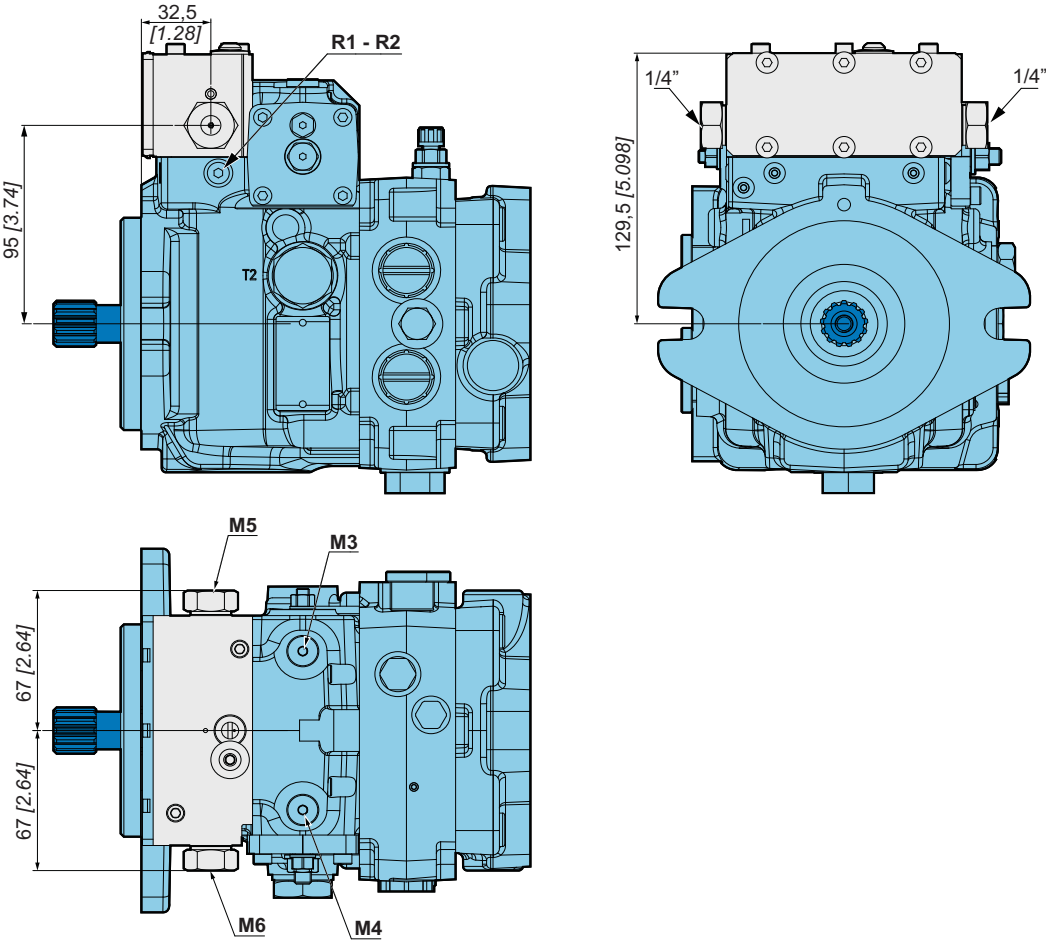
Flow rate determination

Rotation	Pressure	Output	Input
Clockwise (R)	M5	B	A
	M6	A	B
Counter clockwise (L)	M5	A	B
	M6	B	A





Dimensions with control T



Model
Code

Technical
specifications

Operating
Parameters

System design
Parameters

Features

Controls

Options



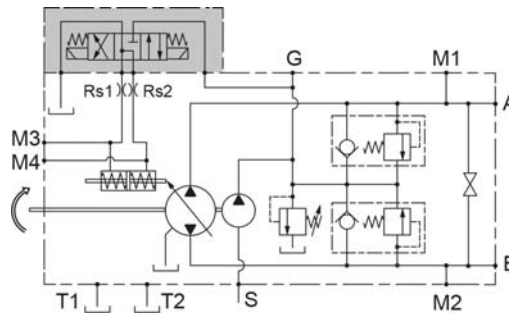
Electrical on-off servo control

Control with return spring

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>

Control function	The change in pump displacement is reached by activation of an ON-OFF electrovalve with closed CETOP 2 connection. If the electrovalve motion is stopped, the pump goes back to neutral position due to the hydraulic servo piston return springs.
Control regulation	The displacement reached is defined by the starting time of the electrovalve and by diameter of restrictors (Rs1 and Rs2) inserted between the electrovalve and the hydraulic servo piston. The pump can be supplied either without electrovalve (B00) or with electrovalve (B12 / B24).

Supply voltage	
Without	B00
12V	B12
24V	B24

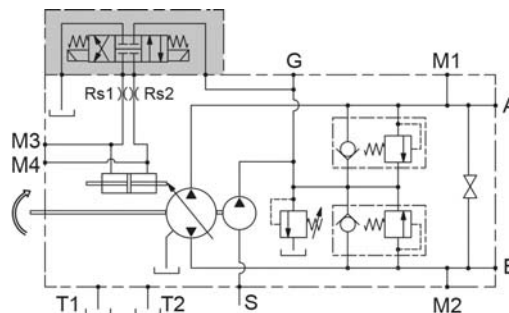


Control without return spring

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>

Control function	The change in pump displacement is reached by activation of an ON-OFF electrovalve with closed CETOP 2 connection. The hydraulic servo piston is without return springs, the pump setting at certain displacement is defined by closed center of electrovalve.
Control regulation	The displacement reached is defined by the starting time of the electrovalve and by diameter of restrictors (Rs1 and Rs2) inserted between the electrovalve and the hydraulic servo piston. The pump can be supplied either without electrovalve (C00) or with electrovalve (C12 / C24).

Supply voltage	
Without	C00
12V	C12
24V	C24



Flow rate determination

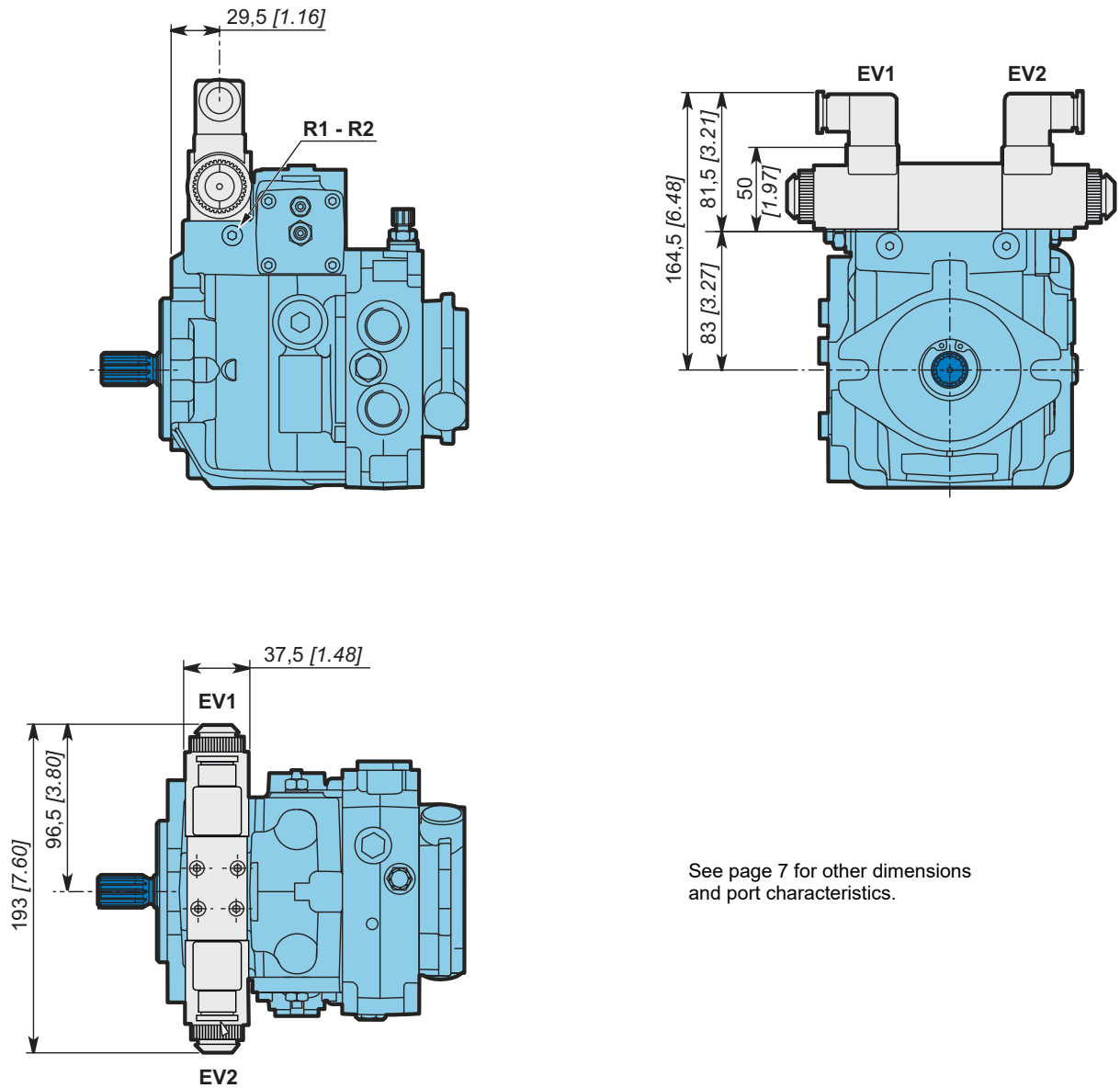
Rotation	Control	Pressure	Output	Input
Clockwise (R)	EV1	A	A	B
	EV2	B	B	A
Counter clockwise (L)	EV1	A	B	A
	EV2	B	A	B



The spring return feature in the control unit is not a safety device.



Dimensions with controls B - C



See page 7 for other dimensions and port characteristics.

Type of connector: Standard DIN 43650 on request Deutsch

Model
Code

Technical
specifications

Operating
Parameters

System design
Parameters

Features

Controls

Options



Electro-proportional servo control



Control function	The variation in pump displacement is reached by current adjustment applied to proportional valve coils. The coils then adjust the pressure of the servo control connected to the hydraulic servo piston. The flow rate direction depends on activated coil.
Control regulation	The reaction time can be controlled by ramps installed on the card and by restrictors (Rs1 and Rs2) positioned between the electrovalves and the hydraulic servo piston.
Automotive function	Electro-proportional servo control combined with ECU and appropriate software can be used for Higher performances automotive control.

Supply voltage

12V	P12
24V	P24

Flow rate determination

Rotation	Control	Output	Input
Clockwise (R)	EP1	B	A
	EP2	A	B
Counter clockwise (L)	EP1	A	B
	EP2	B	A



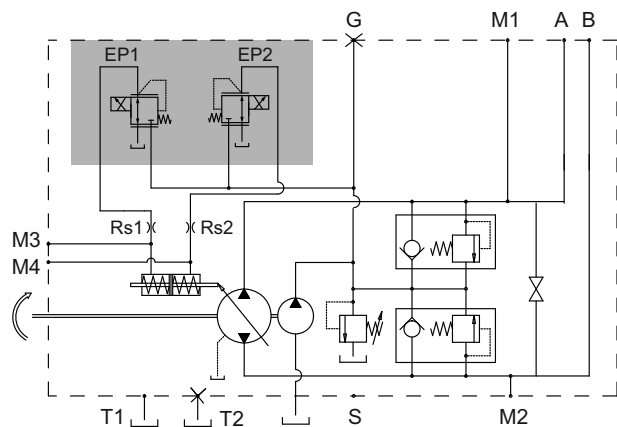
Valve plate timing and regulation curve of proportional valve influence the flow. Contact your Poclain Hydraulics application engineer for further info.

Solenoid specification

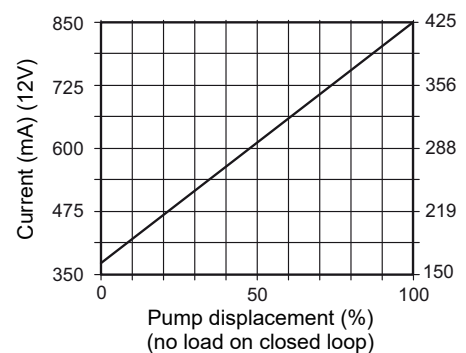
For valve	PWM signal	120 Hz
For coil	12V	resistance 6,5 Ω
	24V	resistance 26,5 Ω



The current must not exceed 1500 mA under 12V and 800 mA under 24V.



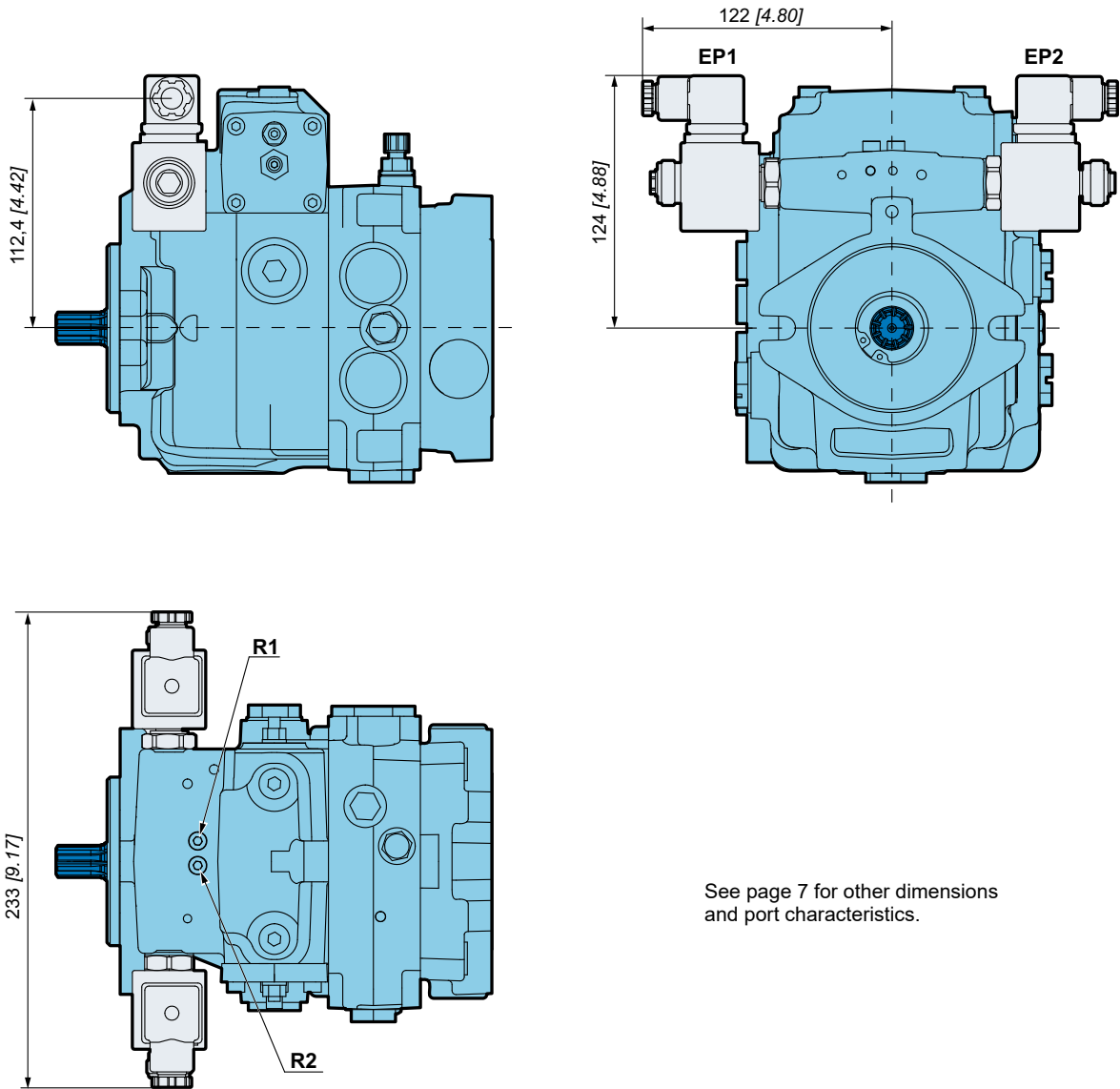
Electrovalve current vs Displacement



The spring return feature in the control unit is not a safety device.



Dimensions with control P



Type of connector: Standard DIN 43650 on request Deutsch

Model Code
Technical specifications
Operating Parameters
System design Parameters
Features
Controls
Options



Electro-proportional servo control with feedback



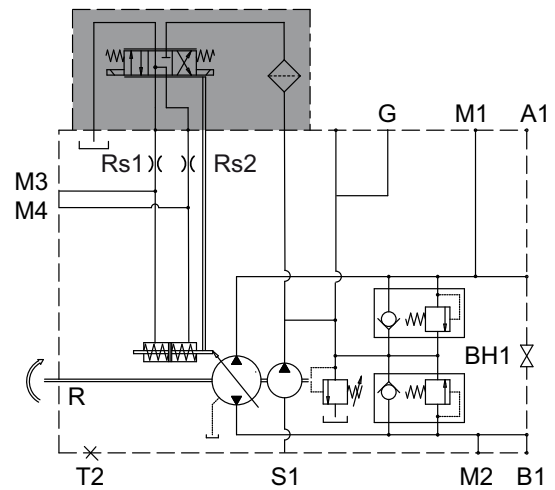
Control function	The variation in pump displacement is reached by current adjustment applied to electro-proportional coils. The coils then adjust the pressure of the servo control. The flow rate direction depends on activated coil.
Control regulation	The reaction time can be controlled by ramps installed on the card and by restrictors (Rs1 and Rs2) inserted between the servo control and the hydraulic servo piston.
Feedback function	The feedback function is reached by a lever that connects the swashplate and the hydraulic servo piston. To avoid sudden accelerations and stoppages, two restrictors (Rs1 and Rs2) are inserted between the servo control and the hydraulic servo piston.

Supply voltage

12V	Q12
24V	Q24

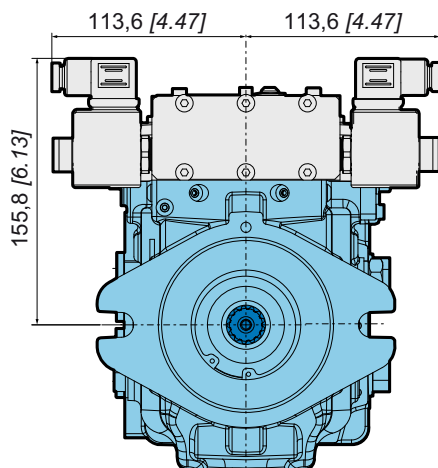
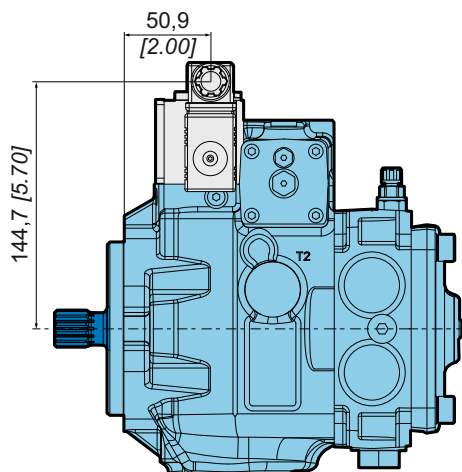
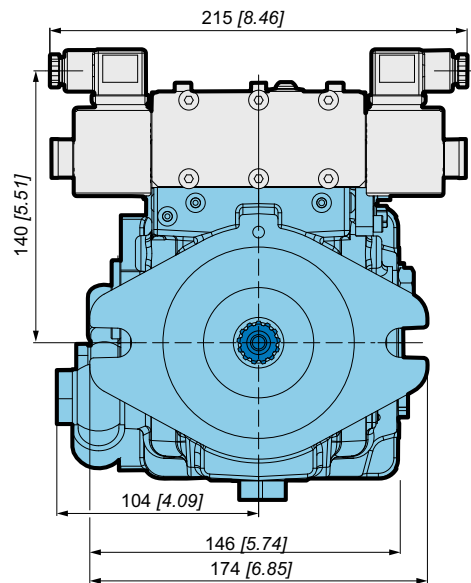
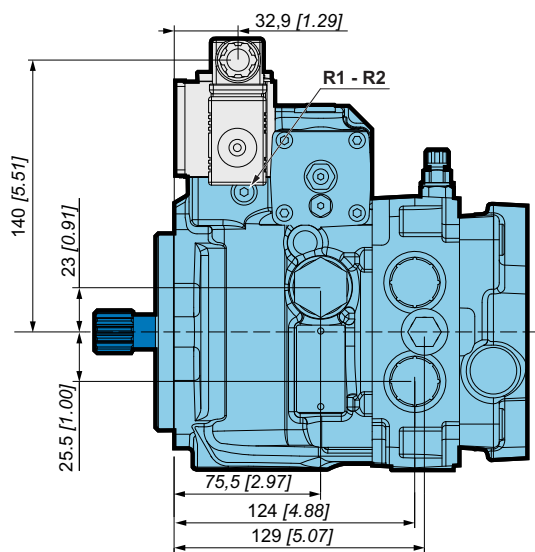
Flow rate determination

Rotation	Control	Output	Input
Clockwise (R)	EP1	B	A
	EP2	A	B
Counter clockwise (L)	EP1	A	B
	EP2	B	A





Dimensions with control Q



Type of connector: Standard DIN 43650 on request Deutsch

Model
Code

Technical
specifications

Operating
Parameters

System design
Parameters

Features

Controls

Options



Hydraulic automotive control



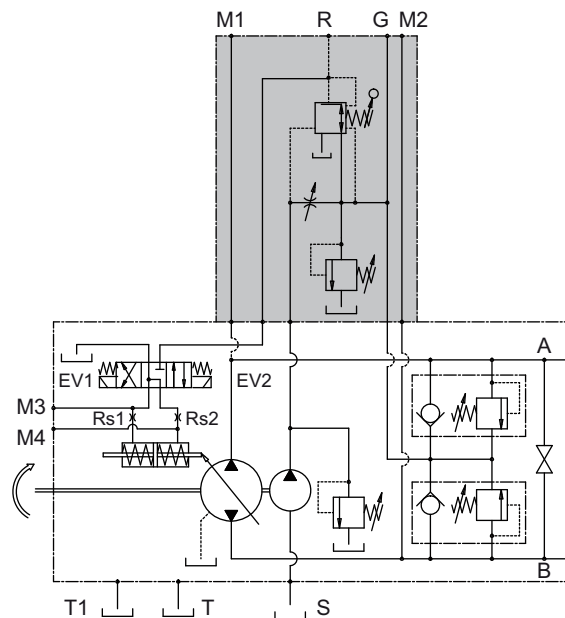
Supply voltage

12V	D12
24V	D24

Control function	The variation in pump displacement is reached by continuous electro-hydraulic valve adjustment. The adjustment is precised by pilot pressure controlled by solenoid control. The pilot pressure increases proportionally to the rotation of the pump. The pump displacement increases corresponding to the higher pilot pressure.
Control regulation	In case the engine is overloaded, the rotation rate decreases and the pilot pressure is reduced causing a pump displacement reduction with a corresponding drop in absorbed power.
Inching function	Inching function is reached by reduction of the pilot pressure, independently of the pump rotation speed. Consequently the pump displacement is reduced.

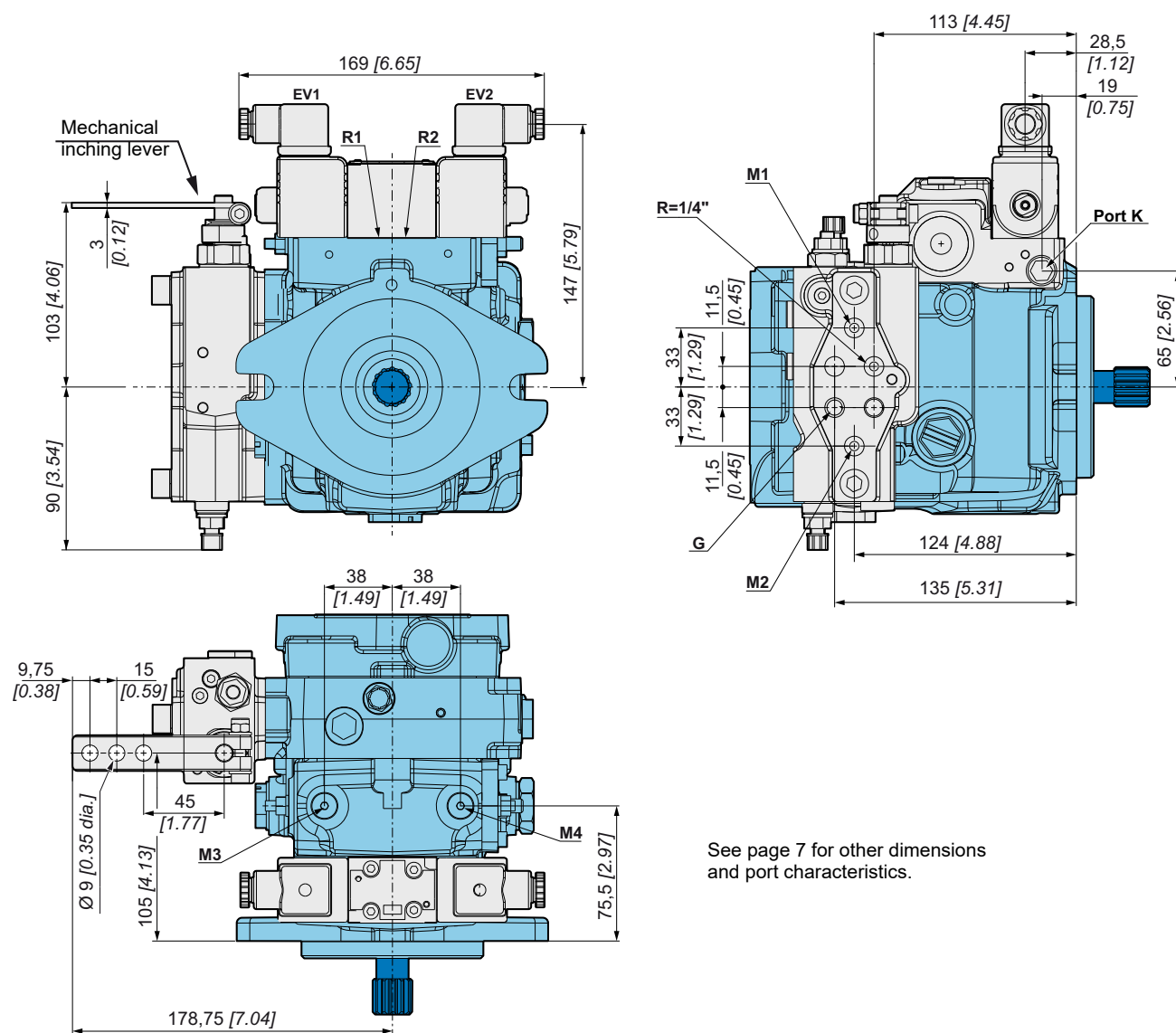
Flow rate determination

Rotation	Control	Output	Input
Clockwise (R)	EV1	B	A
	EV2	A	B
Counter clockwise (L)	EV1	A	B
	EV2	B	A





Dimensions with control D and option IC



See page 7 for other dimensions and port characteristics.

Type of connector: Standard DIN 43650, on request Deutsch



The restrictors R1 and R2 are under the electrical valve.





OPTIONS

Model
Code

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Specifications

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Parameters

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Parameters

Features

Controls

Options

Roller bearing

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
															CR

It is an optional high capacity bearing.

Depending on the characteristics of shaft load, the duty cycle of the application and the expected life time of your application, Roller bearing might be needed.

Consult your Poclain Hydraulics application engineer.

Fluorinated elastomer seals

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
															EV

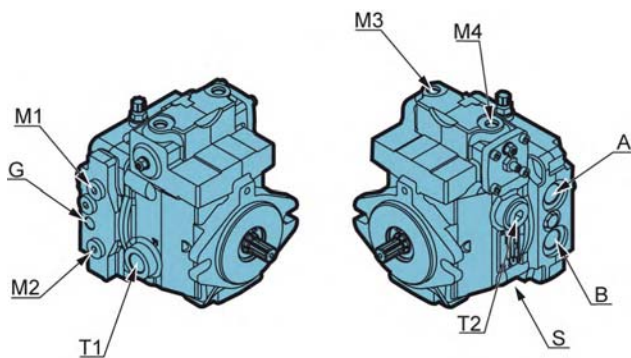
Standard NBR sealing are designed to resist to temperature up to 90°C [194°F] and to HV type oils.

If your application is outside these limits, Fluorinated elastomer seals might be recommended.

Consult your Poclain Hydraulics application engineer.

UNF threads ports

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
															FU



Port	Function	ISO 11926-1 (option FU)
A-B	Services	3/4-16 UNF-2B
G	Auxiliary	7/16-20 UNF-2B
M1/M2	Gauge	7/16-20 UNF-2B
M3/M4	Servo control pilot	7/16-20 UNF-2B
S	Suction	1-1/16-12 UNF-2B
T1/T2	Drain	7/8-14 UNF-2B



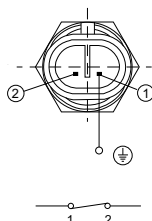
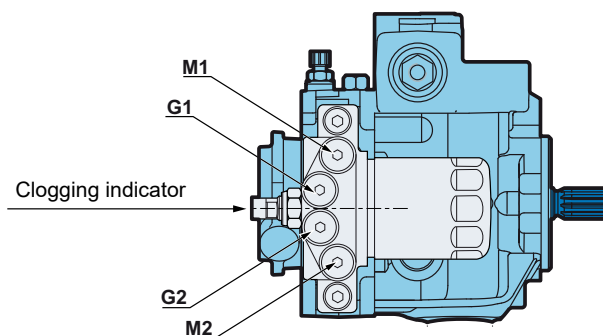
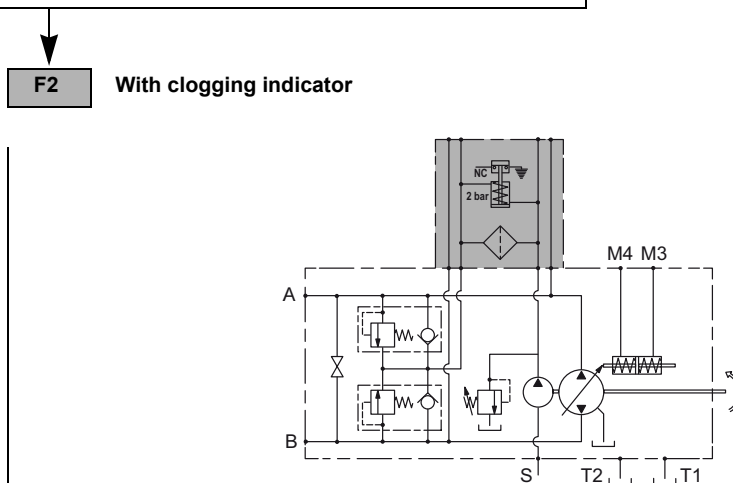
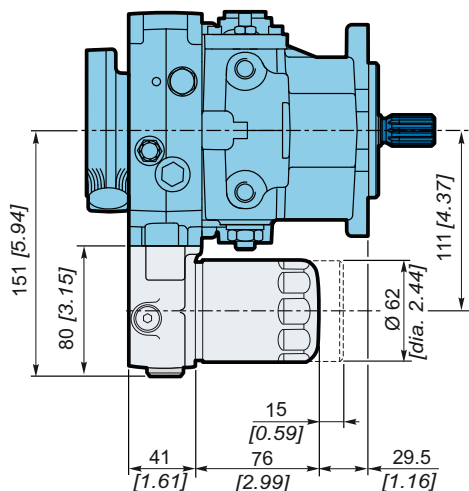
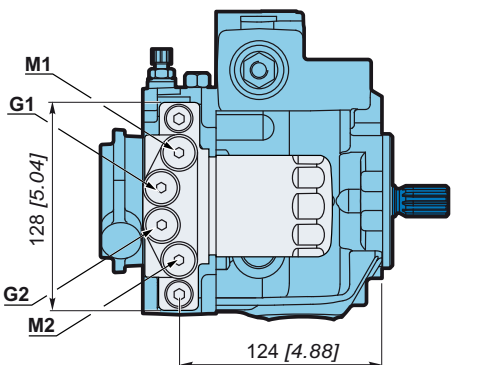
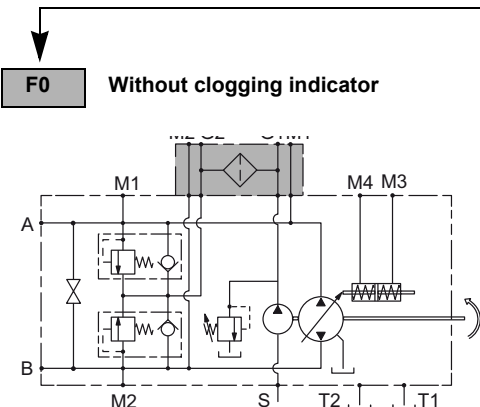
Filter on pressure line

The PM10 pumps can have a pressure filter without clogging indicator (F0) or with clogging indicator (F2). The flow thru the filter is only the flow that entry in the close loop. The filter fitness is of 10 micron.

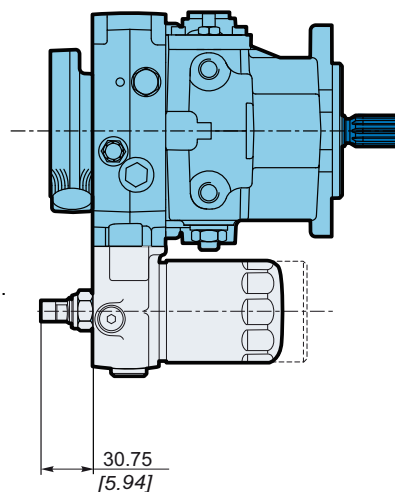
Maximum pressure difference between filter cartridge input and output is 2 bar [29 PSI]. When reaching 2 bar [29 PSI], the cartridge has to be changed.

Max. working pressure: 30 bar [435 PSI].

Tightening torque: 35 Nm [309 in.lbf].



Normally closed contact.
Thread of clogging indicator is internally connected to the ground.



Clogging indicator specification

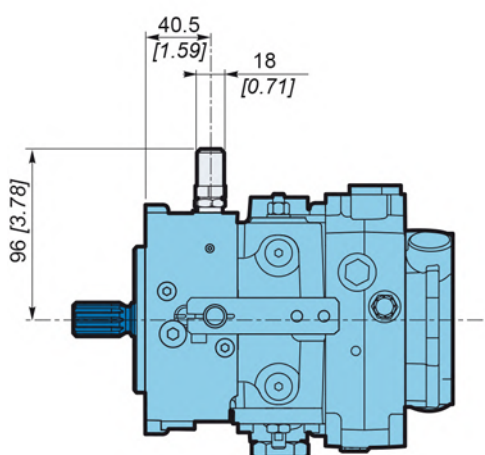
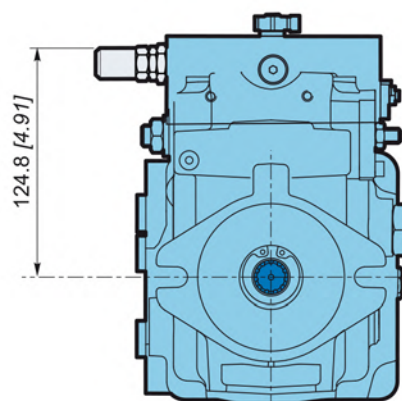
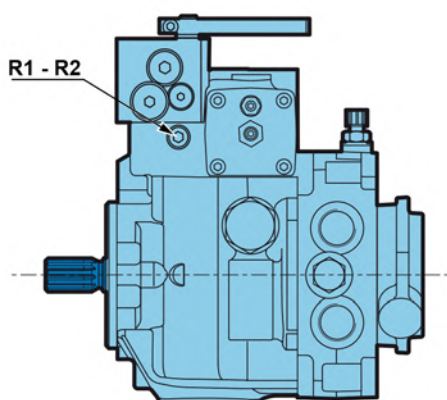
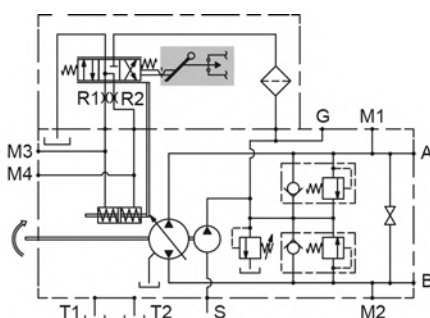
Differential working adjustment	3 ± 0,2 bar [44 ± 3 PSI]
Working temperature	-30°C ~ 110 °C [-22°F ~ 230°F]
Max. vibration level	50 g
Connector type	AMP super seal, 2 way
Current range	0,1-0,2 A max.



Neutral position switch



For the control A it is possible to obtain a micro switch to avoid the start of the engine if the lever of the control is not in center (zero position).



Type of connector: Deutsch DT04-2P

Model
Code

Technical
specifications

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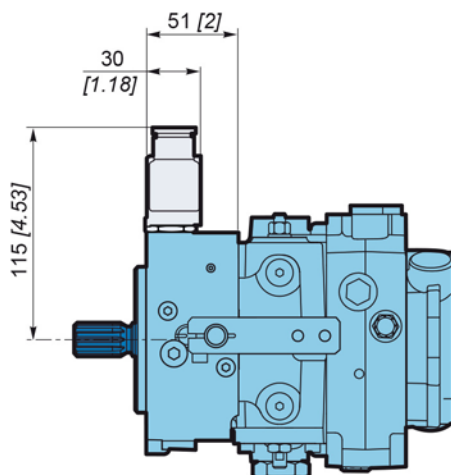
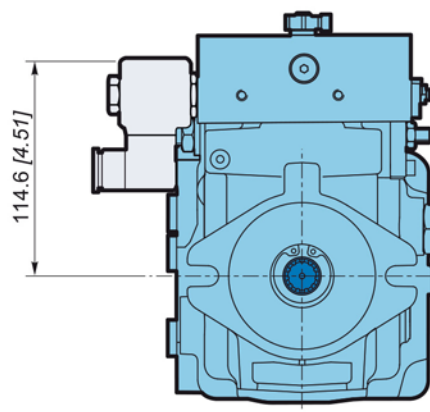
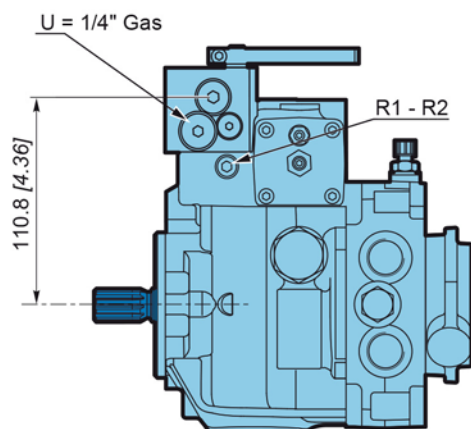
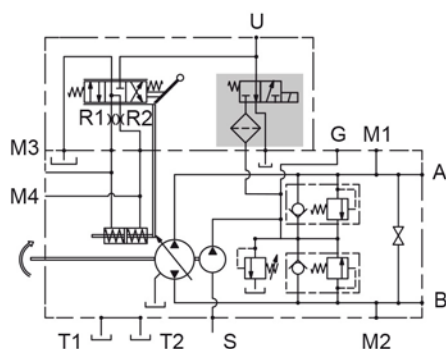
External connections for filter

	1	2	3	4	5	6	7	8	9	10	11	12
P M 1 0	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	F3

Safety valve

	1	2	3	4	5	6	7	8	9	10	11	12
P M 1 0	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	A	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>	VPU

The pump PM10 control A can be provided with a safety valve VPU. Without current, the VPU disconnect the servo control from the charge pressure and engage negative brake.



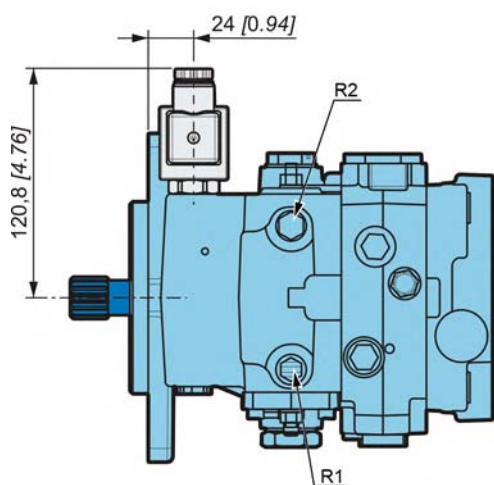
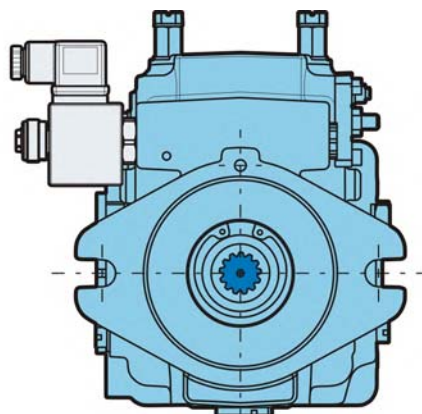
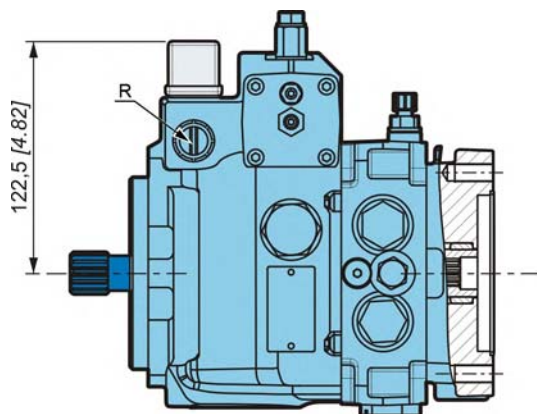
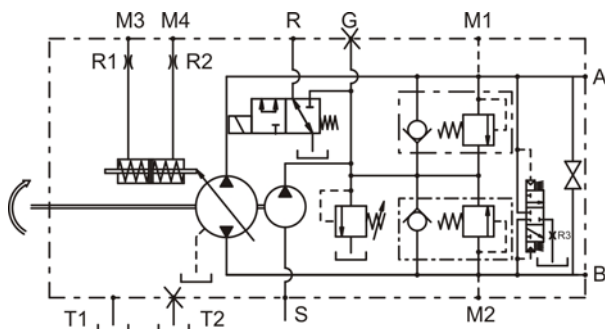
Type of connector: Deutsch DT04-2P



Safety valve



The pump PM10 control S can be provided with a safety valve VPU. Without current, the VPU disconnect the servo control from the charge pressure and engage negative brake.

Model
CodeTechnical
specificationsOperating
ParametersSystem design
Parameters

Features

Controls

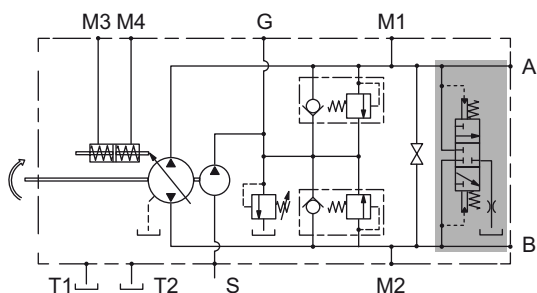
Options



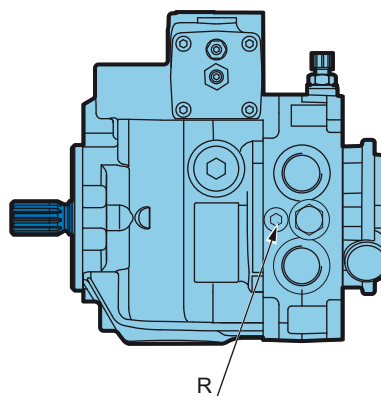
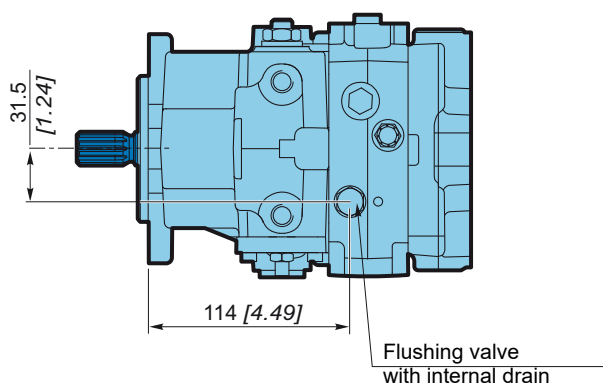
Flushing valve

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
															VS

Inside the pump cover, a flushing valve can be fitted with discharge inside the pump casing by means of a calibrated hole. The flushing valve is useful in case the temperature of the oil in the closed circuit is too high.



Consult your Poclain Hydraulics application engineer for the size of restrictor of flushing valve.



Mechanical inching

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
							D12								IC
							D24								

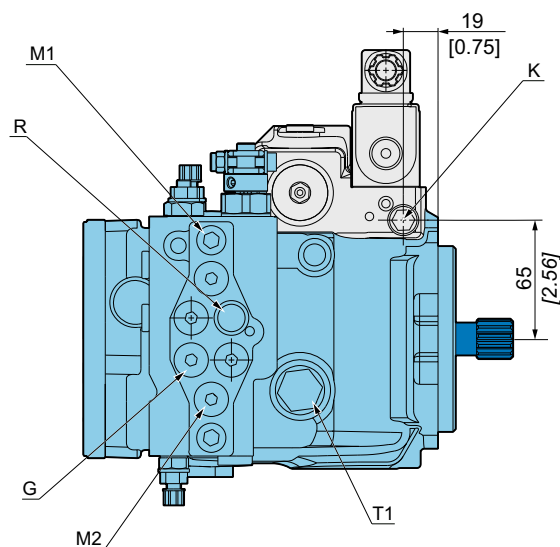
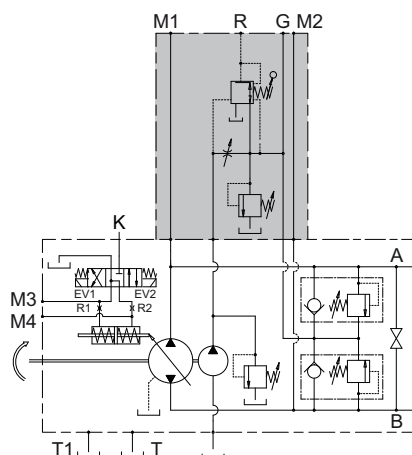
For hydraulic automotive control D. An "Inching" lever is available to reduce the pilot pressure independently of the pump rotation speed. See Hydraulic automotive control D (page 42).



Hydraulic inching

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
			D12												HI
			D24												

For hydraulic automotive control D is available an hydraulic inching HI that consist in a connection K on the pump body to be connect with a pressure reducer valve (for example brake pedal VB002 or VB012). See hydraulic automotive control D (page 42).



Finishing coat

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
															PA

The pumps can be delivered with finishing coat when requested. Standard paint is RAL 9005 (black color).



Consult your Poclain Hydraulics application engineer for other colors of topcoat.

**Special version**

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
															ES

Special execution and incorporate all the note that are not included in the defined options. Example of ES are: valve plate timing, assembly specifications, test conditions, ...



Consult your Poclain Hydraulics application engineer for other possibilities.

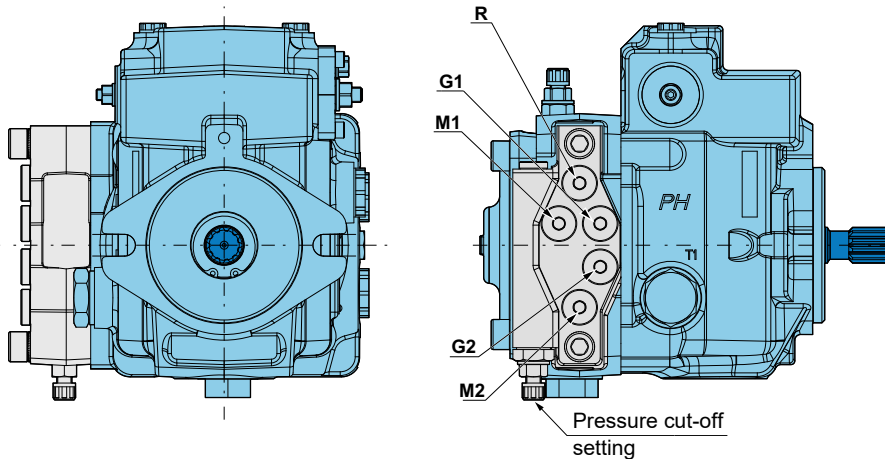
Pressure cut-off valve

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
															LP

The function of the cut-off valve is to avoid that the pump absorbed power exceeds the machine engine power.

The cut-off valve is connected to A and B piston pump's high pressure lines and is usually set at 20 to 30 bar [290 to 435 PSI] lower than the maximum relief valve of the closed circuit.

The cut-off valve acts on the servo control operating pressure in order to reduce the pump displacement and the adsorbed power.



Max deep of the nipples for the connections M1-M2 and G1-G2-R must be 10 mm.

Customized identification plate

P	M	1	0	1	2	3	4	5	6	7	8	9	10	11	12
															DP

It is possible to provide our products with dedicated plate (your part number engraved on the plate) when requested.



This option is available only for minimum volume of 50 pieces.



Consult your Poclain Hydraulics application engineer for other possibilities.



Anti-stall valve

Available for SAE A or SAE B, SD option consists of a block valve (same body as automotive valve) which provide a pressure signal for the servo piston of the pump related to the speed of engine. In case of engine overload and consequent rpm reduction the SD valve reduces the pressure for the servo piston and the pump de-stroke consequentially with an anti-stall effect.



For application of this option please contact your Poclain Hydraulics application engineer.

Model	Code
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Technical specifications

Operating Parameters

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Features

Controls

Options



Poclain Hydraulics reserves the right to make any modifications it deems necessary to the products described in this document without prior notification. The information contained in this document must be confirmed by Poclain Hydraulics before any order is submitted.

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16/11/2022



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